# NASA TECHNICAL **MEMORANDUM**

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SECOND HEATED JETTISON TEST ON THE CENTAUR STANDARD SHROUD

Lewis Research Center `Cleveland, Ohio February 1974

### SECOND HEATED JETTISON TEST ON THE CENTAUR STANDARD SHROUD

Lewis Research Center National Aeronautics and Space Administration Cleveland, Ohio

## ABSTRACT

The second in a planned series of heated jettison tests on the Centaur Standard Shroud was conducted at NASA Plum Brook Station's Space Power Facility on January 16, 1974. The first 250-second portion of the test sequence involved heating the shroud with a specially-built fixture designed to provide a simulation of the heating environment encountered by the shroud during its ascent through the earth's atmosphere. The two heater halves, which were mounted on a rail system, were then retracted. This was followed by the jettison of the two shroud halves into catch nets positioned at 90° to the heater rails. The condition which made this test unique compared to the others in the test series was the alignment of the maximum thermal line with the shroud separation plane. Information on the test hardware, configuration, and sequence is presented. Shroud thermal and deflection data encountered during the heating portion of the test sequence is compared with free-skin design temperatures in various graphical formats.

## INTRODUCTION

The Centaur Standard Shroud protects the payload of the Titan-Centaur launch vehicle during the ascent phase of the flight. To conserve weight, it is jettisoned as early in the flight as possible, while it is still hot from aerodynamic heating. Analysis of the possible flight trajectories and the shroud structure indicated that severe internal stresses could be built up prior to jettison. Calculations of the edge motion of the shroud during jettison indicated that the design clearance between the shroud and the payload could disappear, in the worst case.

An experimental program was conducted in the Space Power Facility at the NASA Lewis Research Center to verify the computer model of the shroud jettison event. The shroud was heated to simulate the expected 280-second flight trajectory then it was jettisoned.

A seven-megawatt, radiant heater was assembled in the vacuum chamber of the Space Power Facility. The heater was programmed to produce the desired temperature distribution with the rlane of symmetry aligned with the separation plane of the shroud. Following the 250-second heating cycle the heater was pulled away to allow the shroud to be jettisoned. A special catch net system was built which allowed one half of the shroud to fall completely free of the launch vehicle while the second half rotated approximately 16° before being caught. The test was performed in a 20-torr environment.

Deflections of the shroud were measured during the heating cycle with straingages and deflectometers. Thermocouples measured the aprlied thermal condition. High-speed motion picture cameras were used to record the motions of the shroud during jettison.

It is the objective of this report to present a brief description of the test hardware, the operation sequence and the results of a preliminary data analysis.

# APPARATUS

The overall arrangement of the test hardware in the Space Power Facility is shown in figure 1. The seven-megawatt heater was built in two halves that rolled on rails perpendicular to the facility rail system. The Centaur shroud was mounted on a Titan-Centaur interstage adapter. Its location in the 100-foot diameter chamber was chosen to allow one half of the shroud to fall completely free of the hinges before being caught in the net. The other half (the one with the dome) was caught after only 16° of rotation. The Centaur tanks were not used in this test because of the unnecessary complexity they would have added. In their place, a special structure was mounted on the

interstage adapter that allowed access to the inside of the shroud and supported the flight truss adapter, equipment module and a simulated pay-load. A photograph of the internal structure is shown in figure 2.

Catch Nets. Special catch-nets were constructed using a high temperature synthetic webbing supported by 6-inch-diameter aluminum pipe frames. The photograph in figure 3 shows the full-jettison catch net in position. The full jettison net frame was supported by cables attached to 10 disk brakes (five on each side) which served to absorb the energy imparted to that shroud half. The catch system was pre-tested using a model of a shroud half to insure that it would function properly without damage to the shroud.

Heater. The heater was designed to duplicate, in time and temperature, the condition expected in the ascent phase of the flight. The heater contained 5910 tungsten filament lamps inside a highly polished aluminum reflector. A detailed thermal analysis (the approach used is described in ref. 1) and extensive small scale tests were performed to verify the design concepts. The heater was divided into 18 separate control zones, 11 in the cylindrical section and 7 in the biconic section, to provide the proper circumferential temperature profiles. In addition the spacing of the lamps was varied within each zone to control the vertical distribution of heat. Because the desired temperature profiles were symmetrical around the maximum heat line, the 18 control zones were further divided into mirror-image half-zones (one on each side of the plane of symmetry). The arrangement of the control and mirror image half-zones is shown in figure 4. The maximum heat line for this test was at an azimuth of 0° (0° from the shroud separation plane).

Control Systems. Each control half-zone and its mirror image was powered by a separate SCR controller. The 18 controllers were programmed individually to reproduce the expected temperature vs time curve for their respective control zones. Abort limits were established to insure that the test would not proceed if any control half-zone or mirror image half-zone deviated more than a prescribed amount from its desired temperature curve.

A PDP-8 mini computer was used to conduct the test because of critical timing of events necessary. The sequence of events for this test is presented in Table 1.

# INSTRUMENTATION

Thermocouples, straingages, deflectometers, and high-speed motion picture cameras were used to measure the performance of the shroud during the test. Digital data were recorded every second during the test, using an XDS 930 computer. FM analog recordings were also obtained of selected parameters. The coordinate system used to define the location of sensors on the shroud is shown in figure 5 and 6. The cylindrical section of the Centaur shroud is a complex structure composed of a corrugated outer skin bonded to a smooth inner skin supported by circumferential "Z" rings. A sketch of the structure and a typical free skin thermocuple installation is shown in figure 7. Free skin thermocouples were located as far from structural masses as possible to provide the best possible measurement of the thermal environment. Free skin ther-

mocouples at station 2626 in the cylindrical part of the shroud, and at station 2724 in the conic part were used to provide temperature feed back to the power controllers.

### RESULTS

The heated jettisen test was conducted on January 16, 1974, at an ambient pressure of 20 torr. The heater was programmed to produce the desired temperature distribution with the plane of symmetry displaced 0° from the shroud separation plane (see figure 4). The light half of the shroud (the one without the dome) was fully jettisoned and fell free of the hinges into the horizontal net. The other half was caught after only 16° of rotation.

Time histories of the control thermocouple readings are presented in figure 8 for the 18 control half-zones and the 18 mirror image half-zones. Included also on this figure are the desired temperature histories. Comparison of desired and measured temperatures shows that excellent agreement was obtained. The greatest deviation was in zone 10 where a 10° F deviation was observed in the mirror image half-zone. The instantaneous power applied to the shroud varies according to the slope of the desired temperature curve. The measured power applied to zone 1 is presented as an illustration in figure 9. The initial peaking power occured because the shroud was cooler than the set point when the heating cycle started. The power increased gradually, following the desired temperature curve. Very little power was needed near the end of the cycle because the required temperature was actually decreasing slowly.

Circumferential temperature profiles are presented in figure 10 at several stations and for several times during the heating cycle. Also shown are the desired temperature profiles. Comparison of the two indicates that very good agreement was obtained everywhere except at the top of the biconic section and at station 2250. The thermocouples at station 2250 are very close to the aft seal bulkhead which probably accounts for their low readings. This is supported by the fact that a thermocouple at station 2469, where the shroud skin thickness and the lamp spacing were the same as at station 2250, agreed very well with the desired curve. These deviations were observed early during the heater checkout tests and were deemed acceptable.

The shroud deflected during the heating cycle because of temperature gradients of as much as 90° F in the "Z" rings. The deflection was measured with potentiometer type deflectometers. Circumferential plots of their readings at several stations and at several times during the test are presented in figure 11. For reference purposes the desired temperature curves are also included in figure 11. Examination of these data indicated that the shroud assumed a pinched cross section with the narrow part at the shroud separation plane. The tendency for the shroud to pinch this way is resisted by the joint between the two halves. Consequently, when the shroud is separated the first motion is expected to be inward. Edge motions of the shroud were recorded by high speed motion picture cameras. At the time of this writing, the cameras data had not been fully analyzed and could not be included. How-

ever, it was observed that the first motion of the shroud edges was inward (about  $3\frac{1}{2}$  inches) toward the payload. In addition to the cameras some short wooden sticks mounted in foam blocks were installed to indicated invasion of the payload envelope by the shroud. First inspection indicated that the payload envelope was not invaded.

Examination of the camera data also revealed small objects (tape-like in appearance) being blown off at high velocity from the Super\*Zip near the hinge in Quadrant II (at approximately 100° azimuth).

#### CONCLUSIONS

A successful heated jettison test of the Centaur Shroud was performed in the Space Power Facility on January 16, 1974. The shroud was heated to the desired thermal condition with the axis of symmetry displaced 0° from the separation plane. Initial observations indicated that the shroud's first motion was inward but that it did not invade the payload envelope.

#### REFERENCE

1. Hemminger, Joseph A.: Computer Simulation of Temperatures on the Centaur Standard Shroud During Heated Jettison Tests. Paper presented at the Seventh Space Simulation Conference, Los Angles, Calif., Nov. 12-14, 1973.

Table 1. SEQUENCE OF EVENTS

Event	Test Time
Start all recorders. Verify recorder start. Start heating cycle. Verify heaters started. Color movie lights on. Color cameras on. Safe zone heaters. Start heater retract. Turn on movie lights. Check heater clear. Start cameras. Arm seal pyro. Fire seal pyro. Verify seal pyro fired. Arm instrument disconnects. Fire instrument disconnects. Verify instrument disconnects fired. Arm Super*Zip for shroud jettison. Fire Super*Zip. Safe all systems. Stop all recorders.	-10 - 8 0 50 to 80 160 165 250 270 272 272 272 272 275 276 277 278 280 280 281 295 300

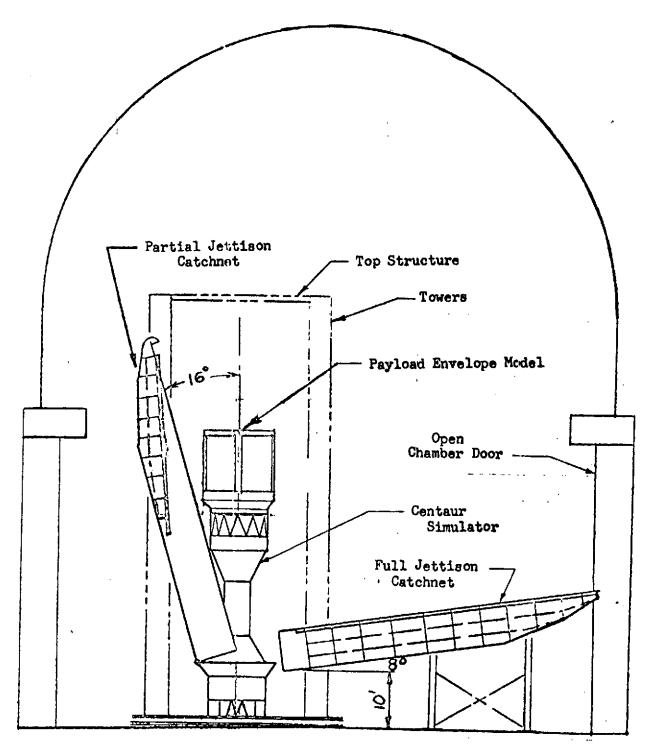


Figure 1. CSS in jettisoned position (view looking north).

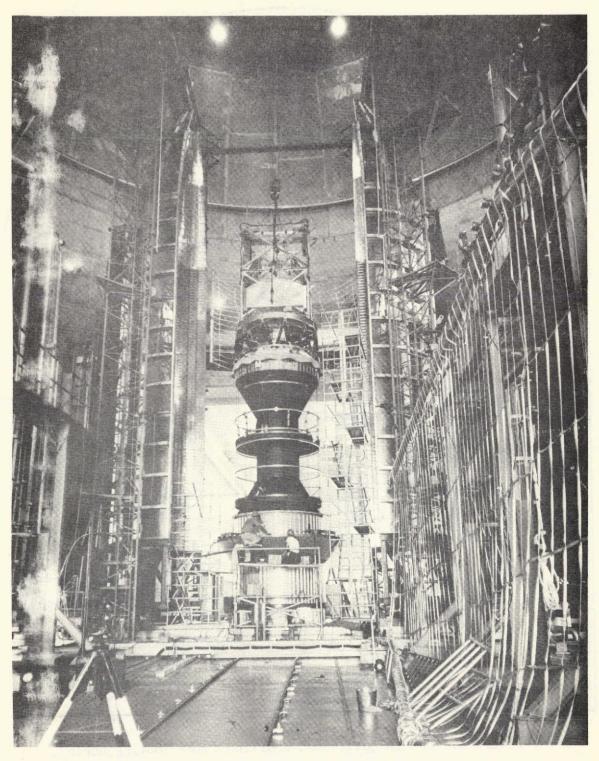


Figure 2. Photograph of internal structure.

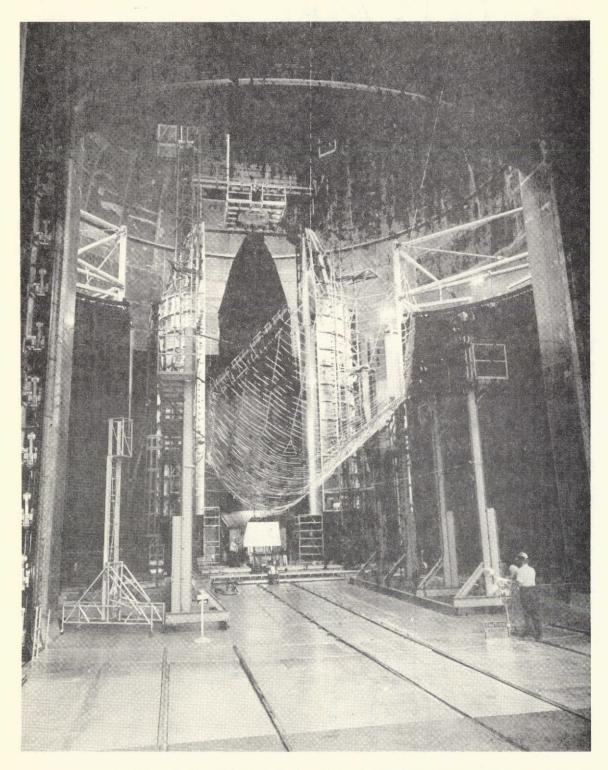


Figure 3. Photograph of test installation.

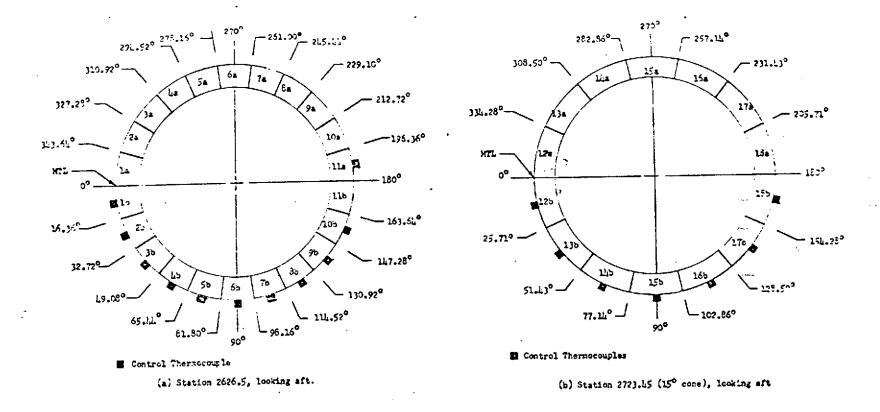


Figure 4. Heater control zones and control thermocouple locations.

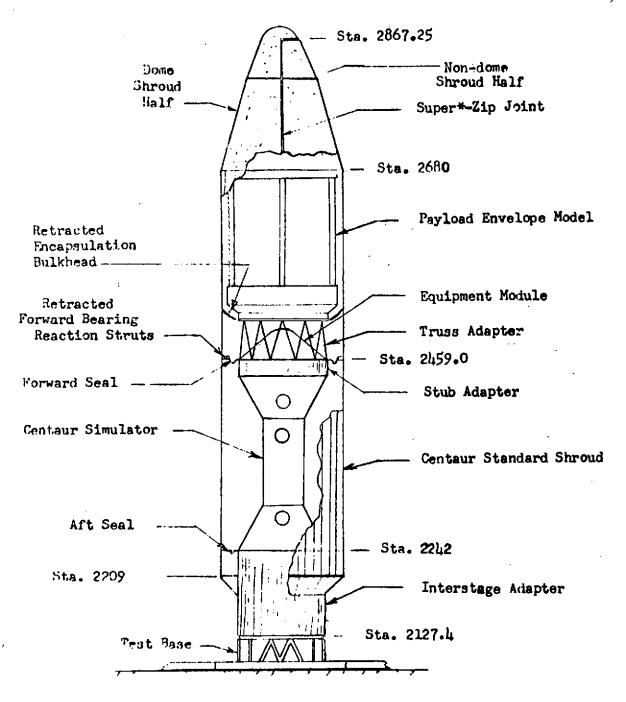


Figure 5. Axial coordinate notation - station locations (inches from datum).

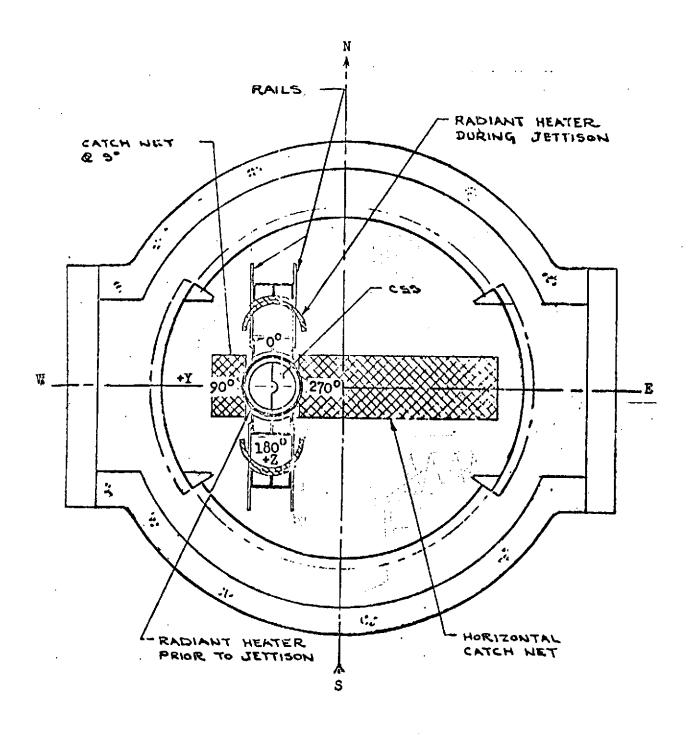
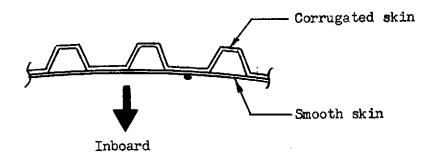
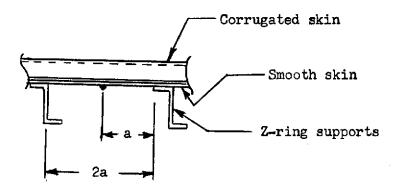


Figure 6. Angular coordinate notation - azimuth (plan view, looking down).



Note: thermocouple junctions located between "weldbonded" panels spot welds.



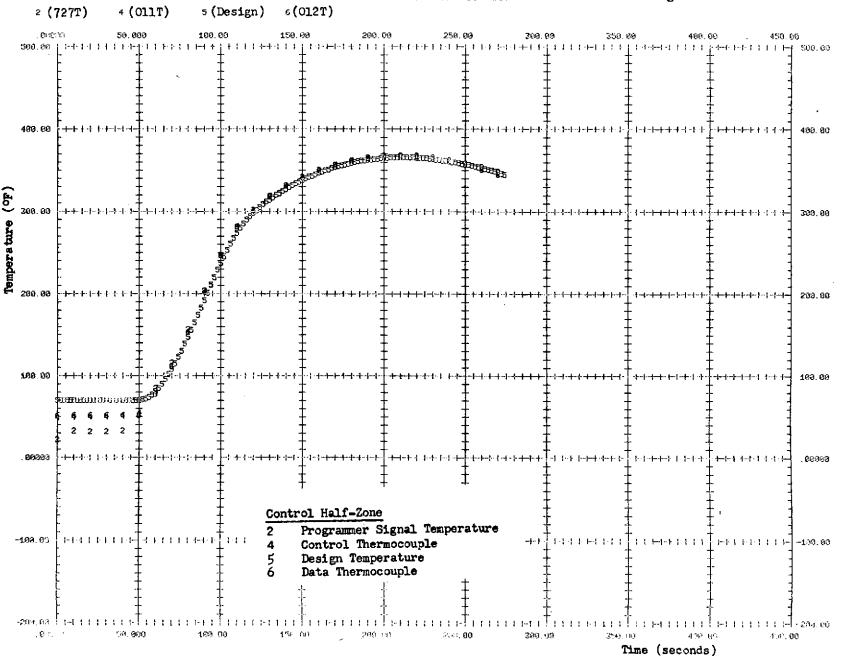
Typical mounting on cylindrical section

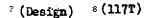
Figure 7. Free-skin thermocouple locations.

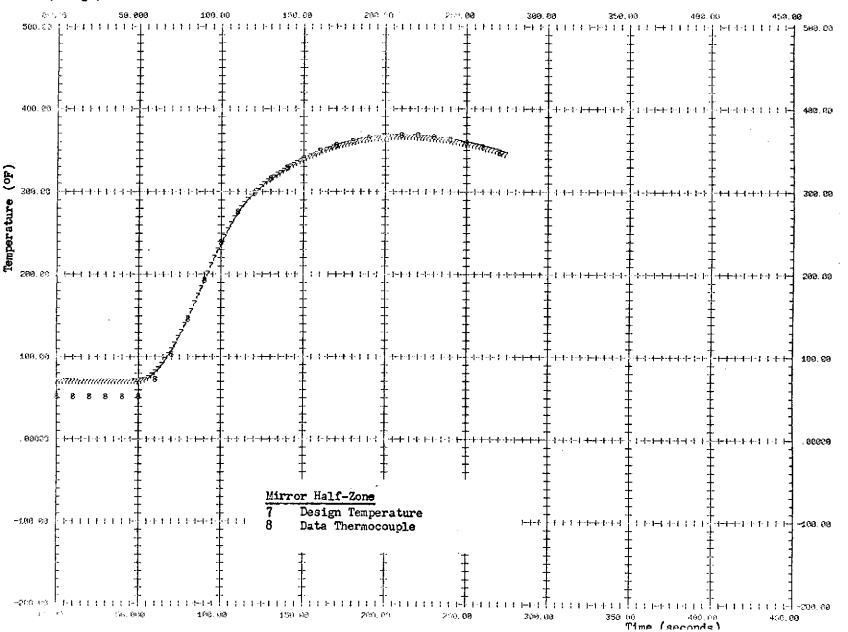
Figures 8.1 thru 8.36.

Design temperature and thermocouple histories at heating zone control locations on CSS.

Figure 8.1



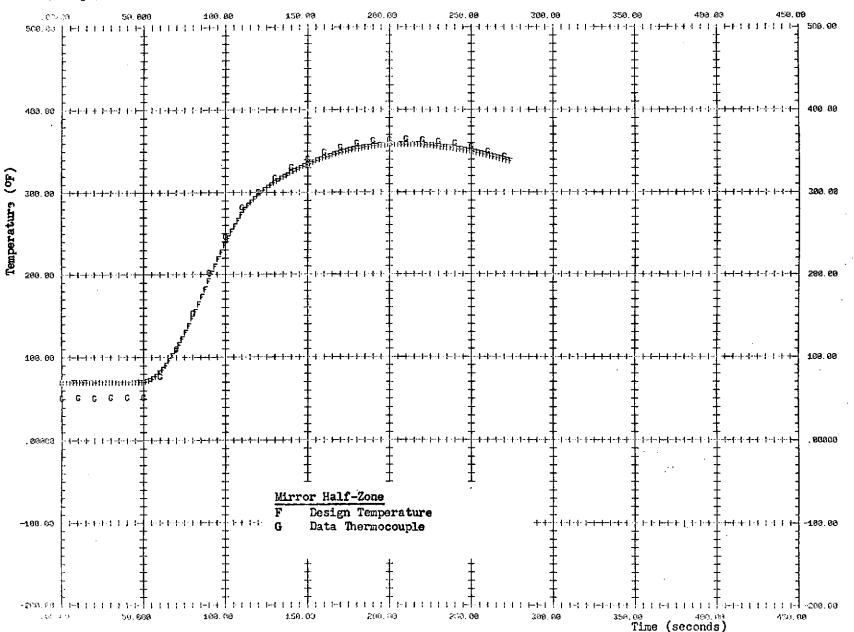




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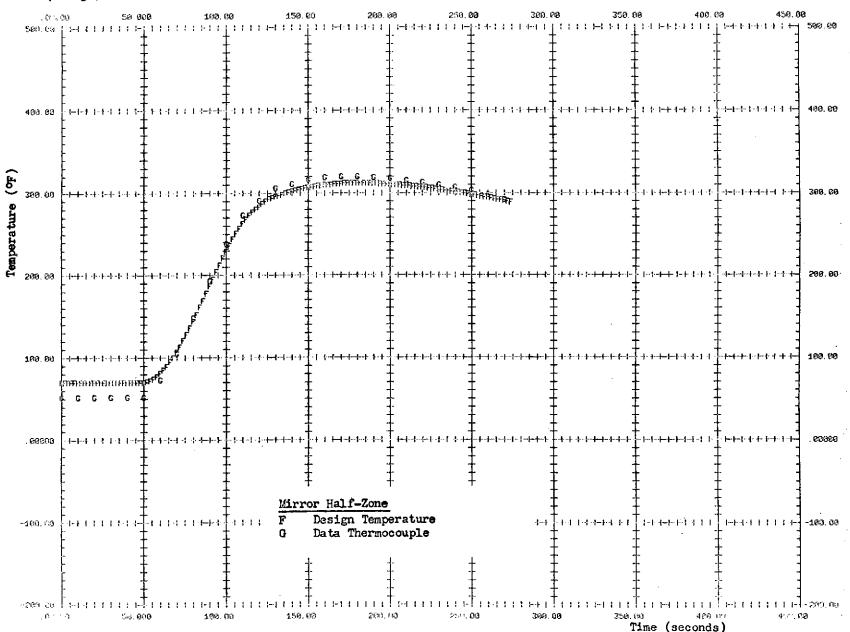
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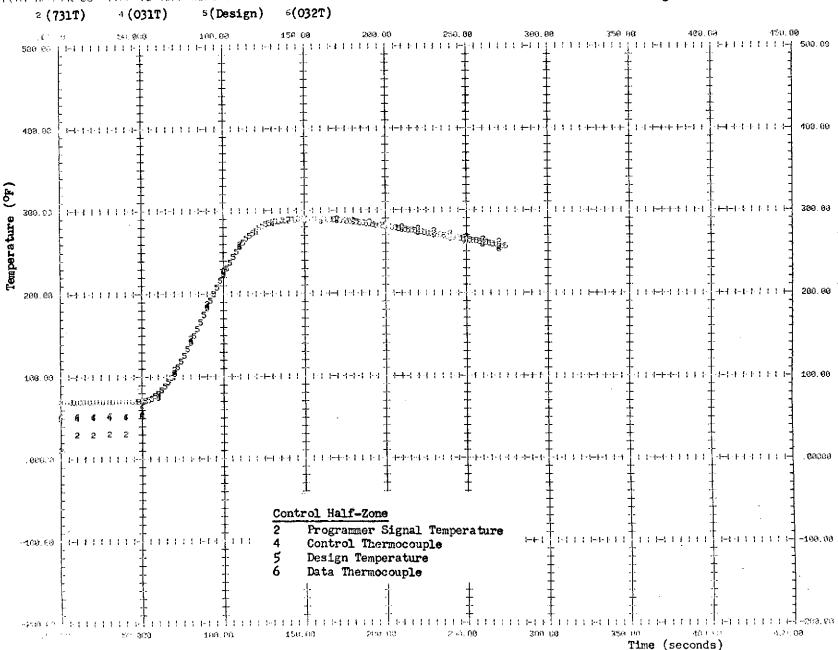
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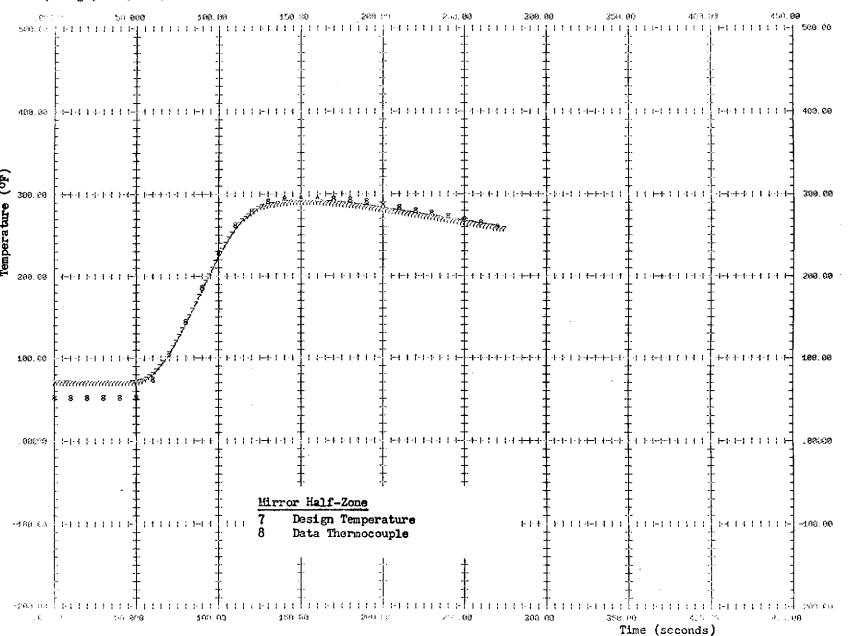
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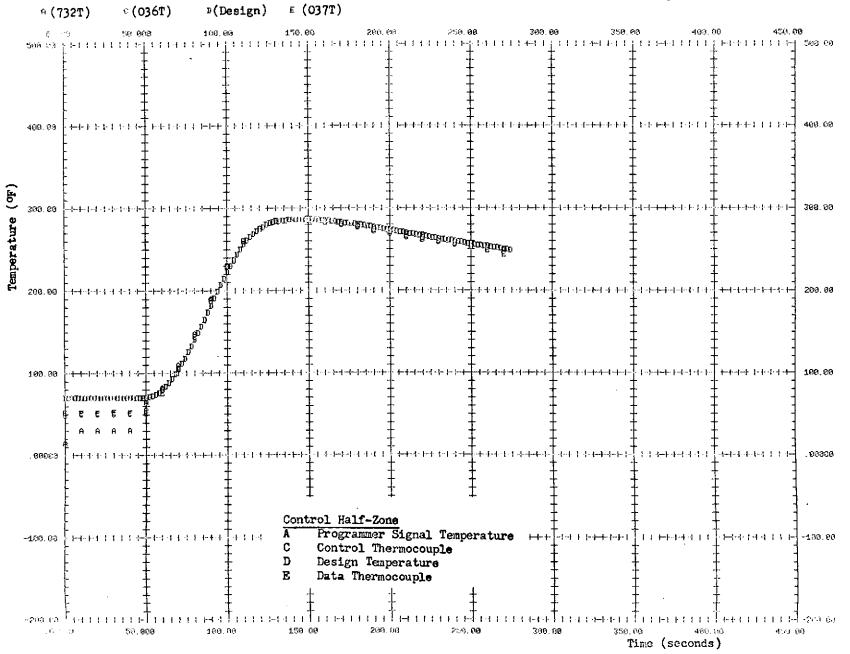


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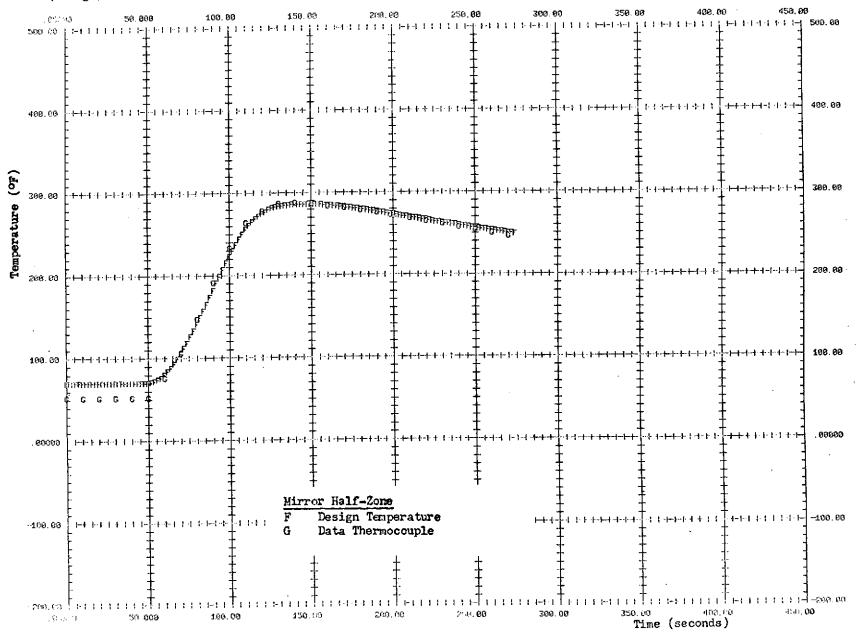


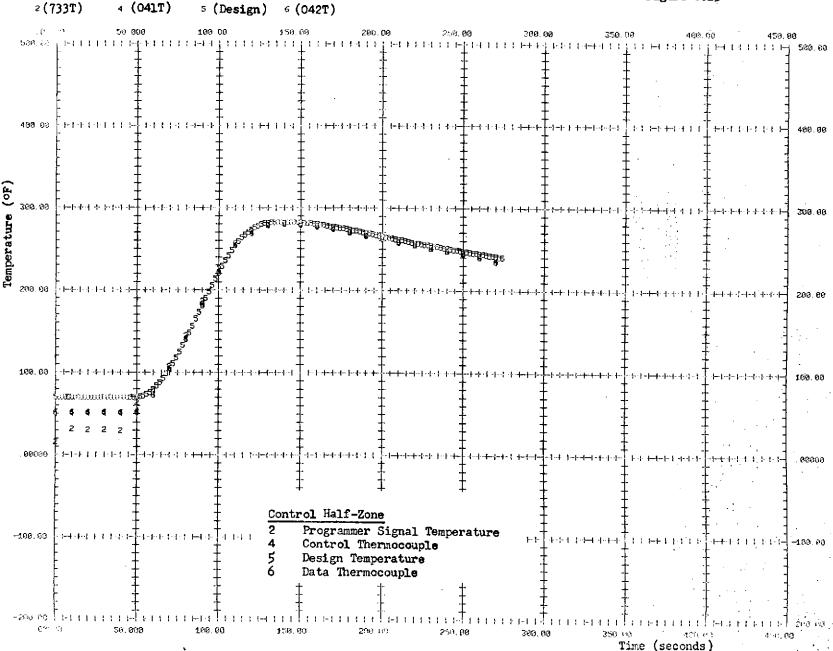
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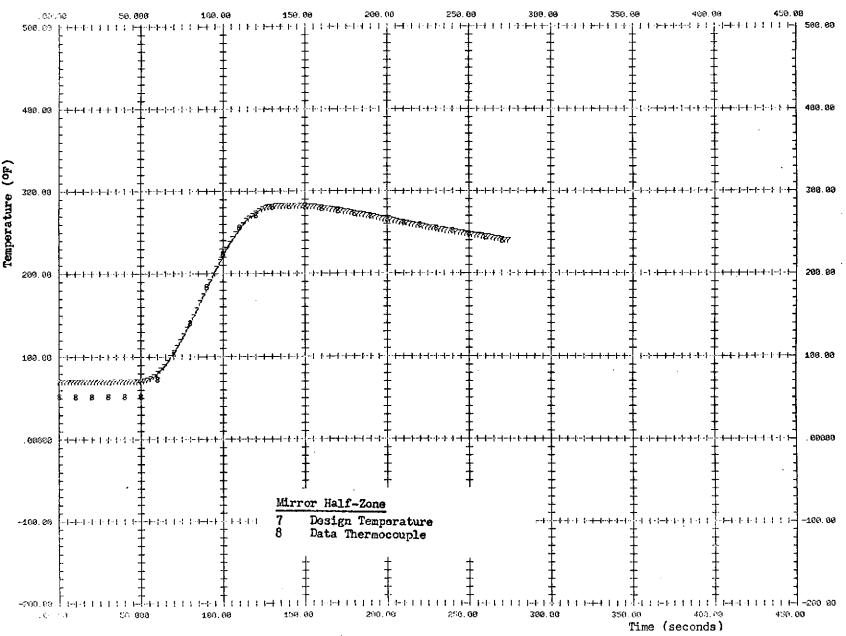


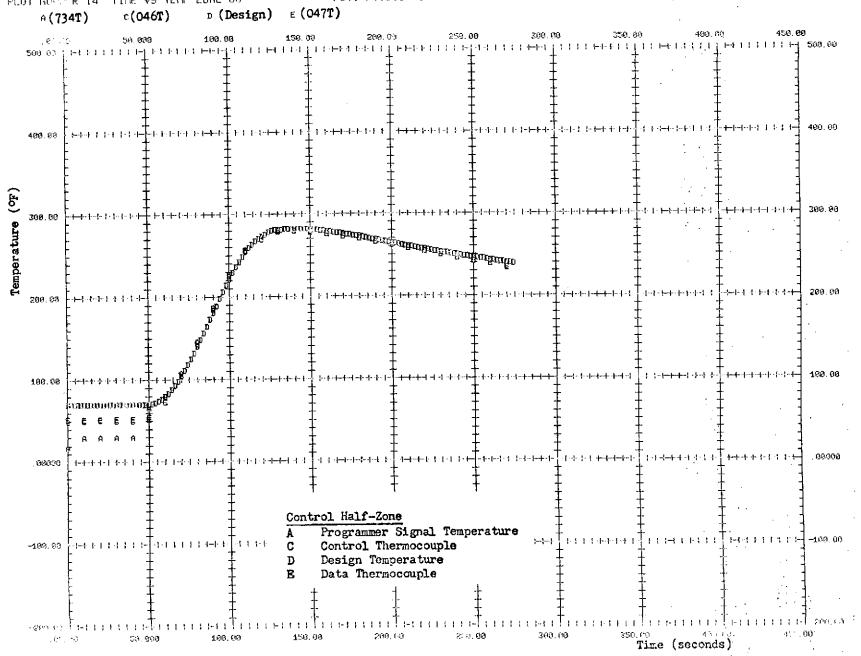




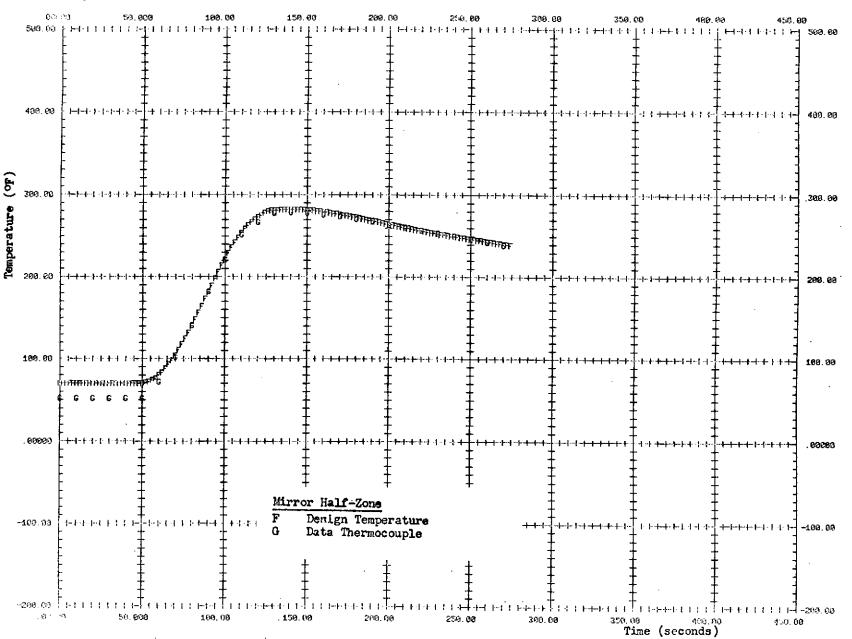


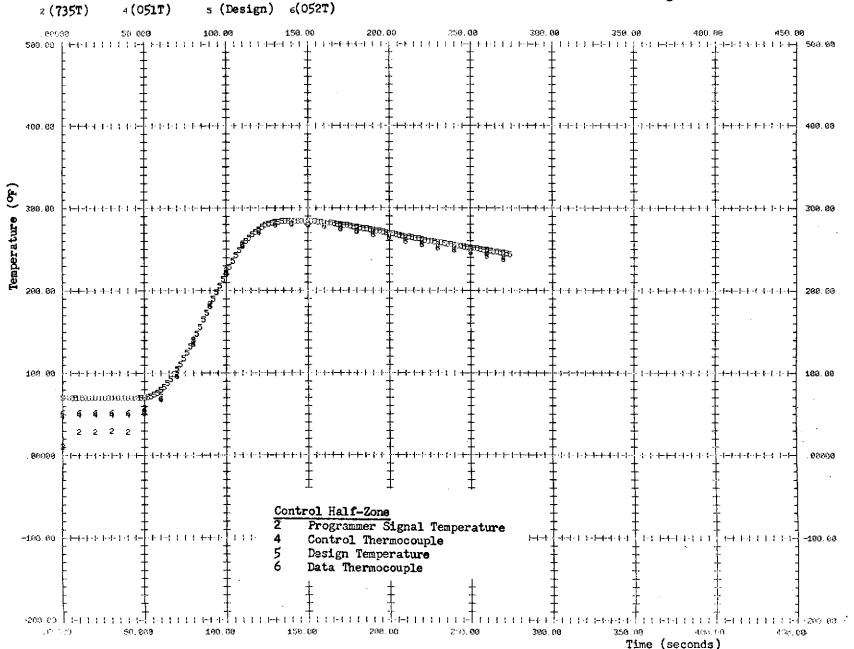
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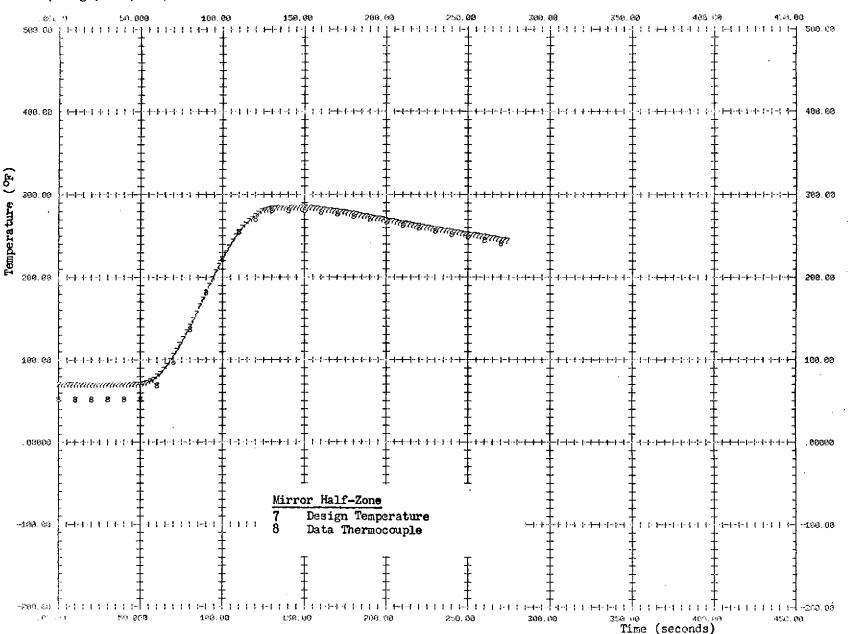


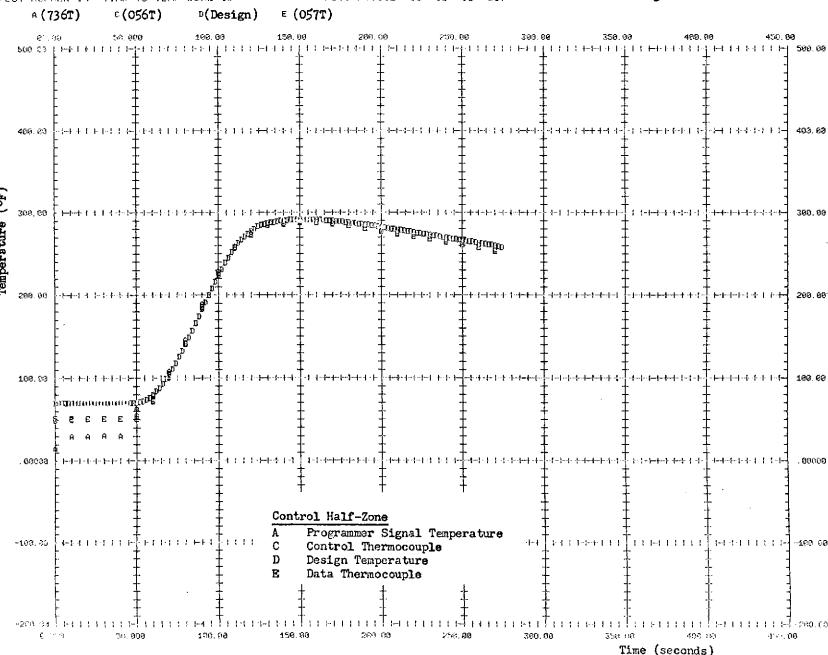
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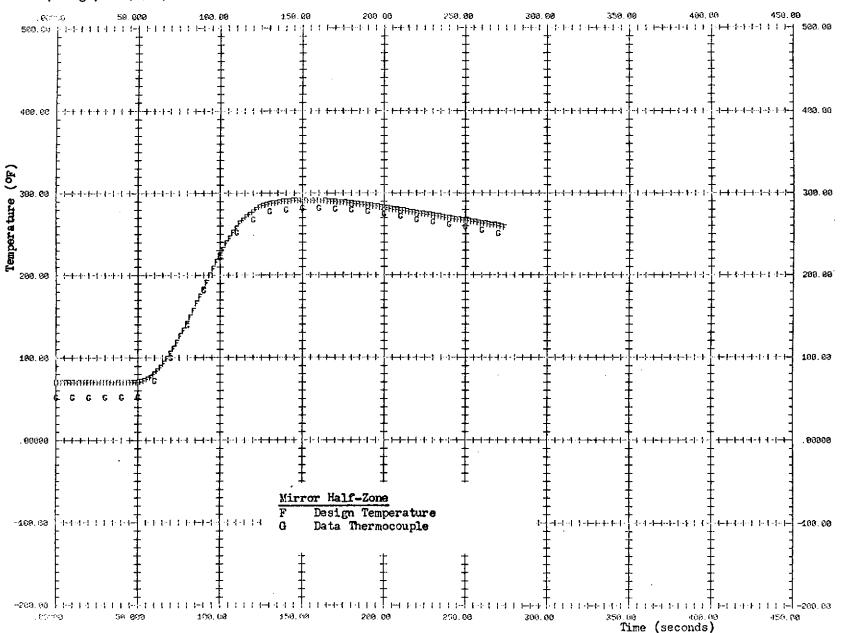


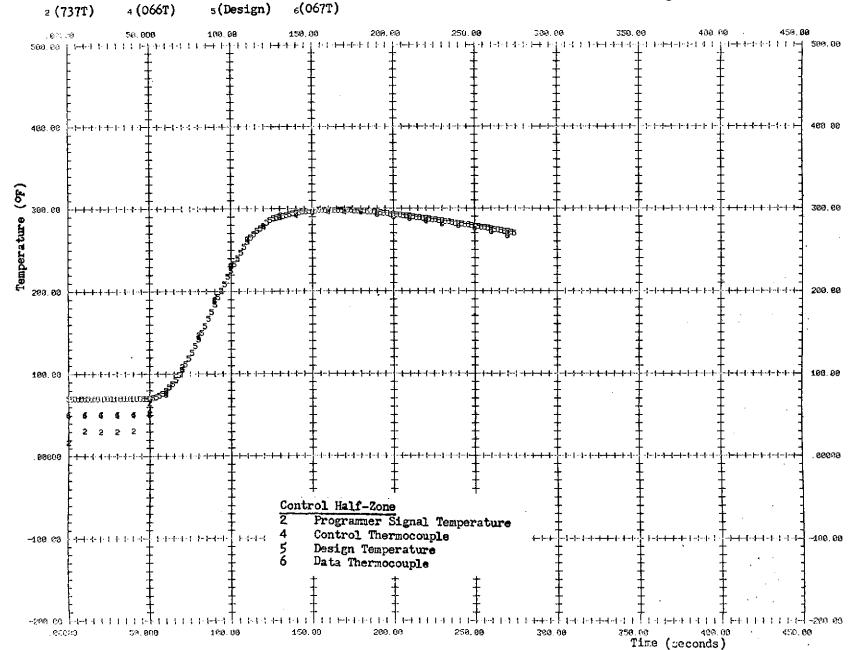
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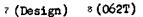


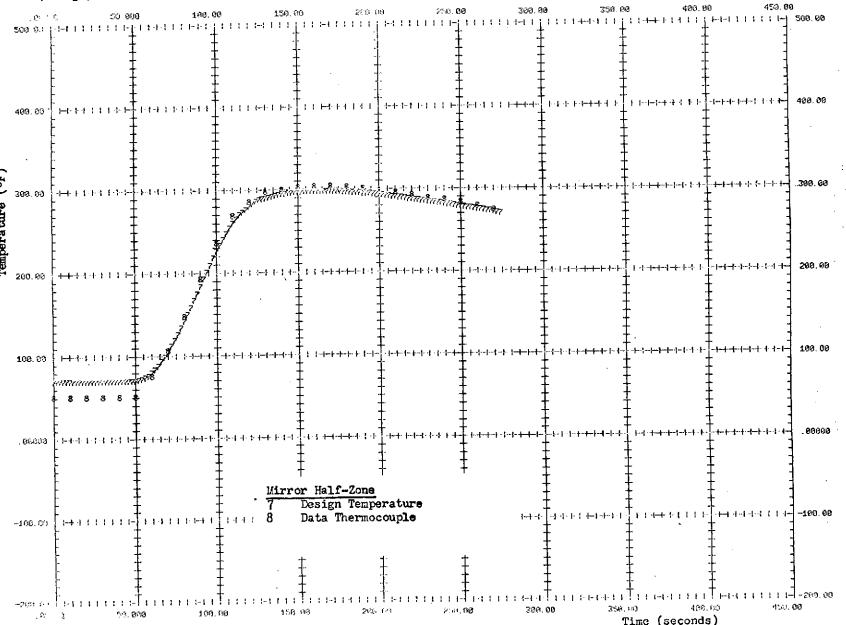


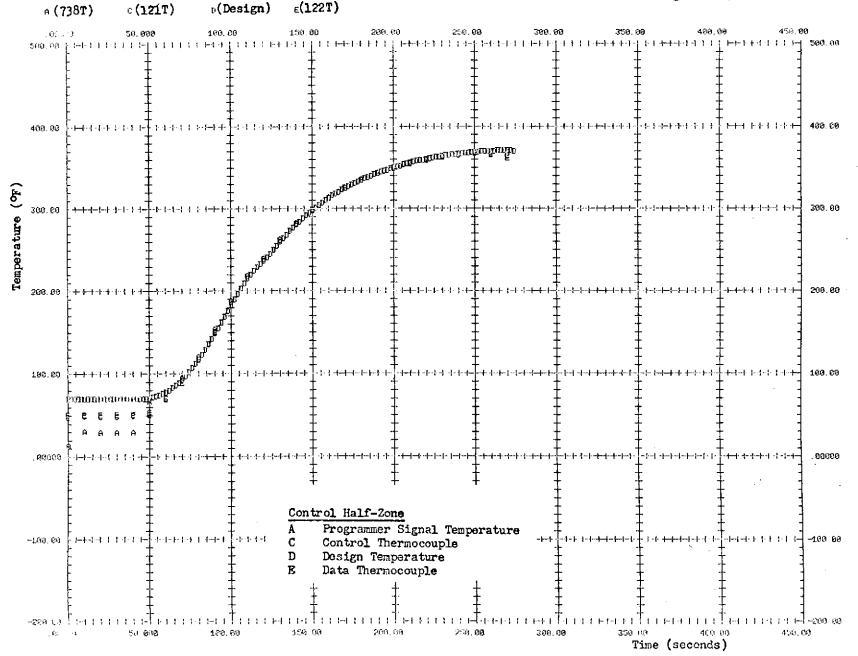
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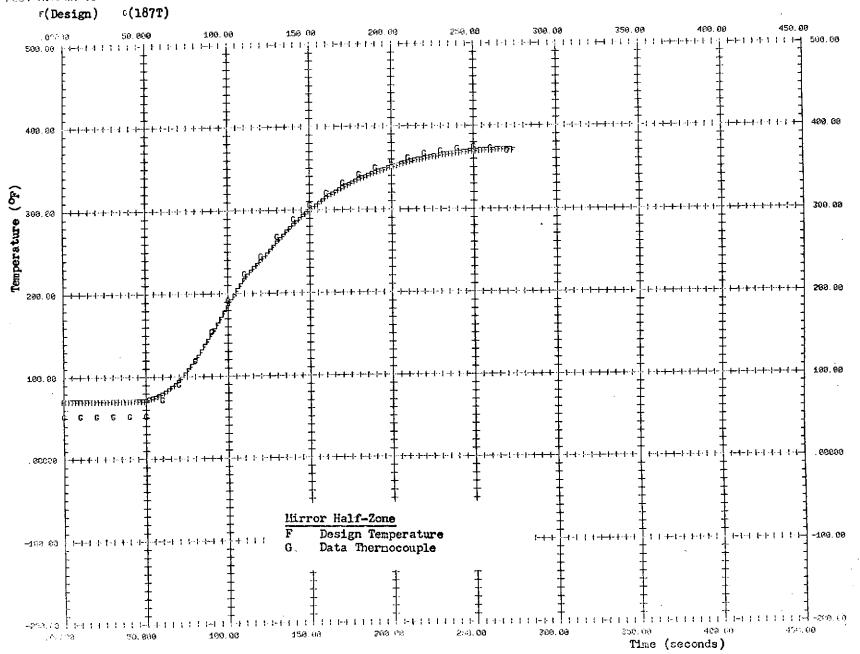


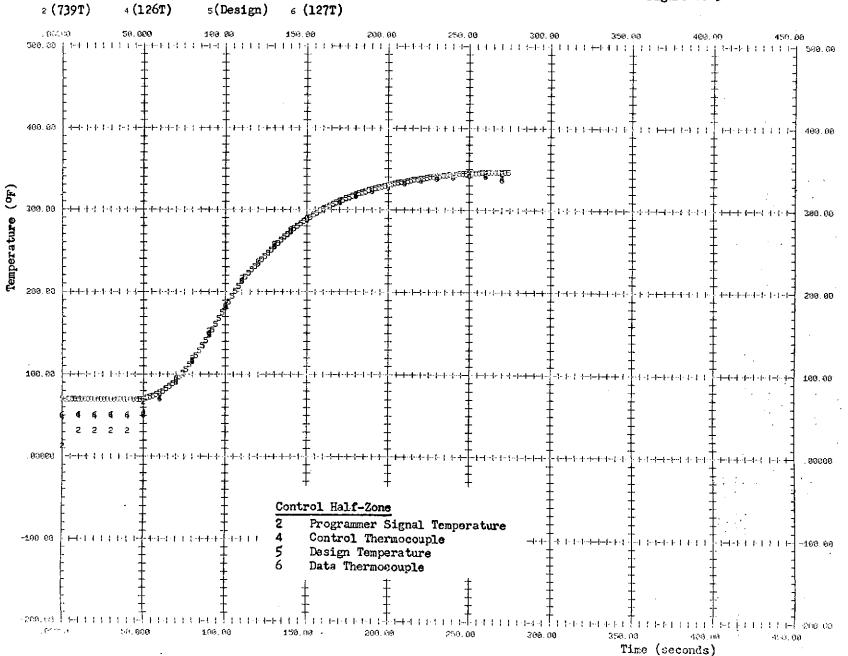




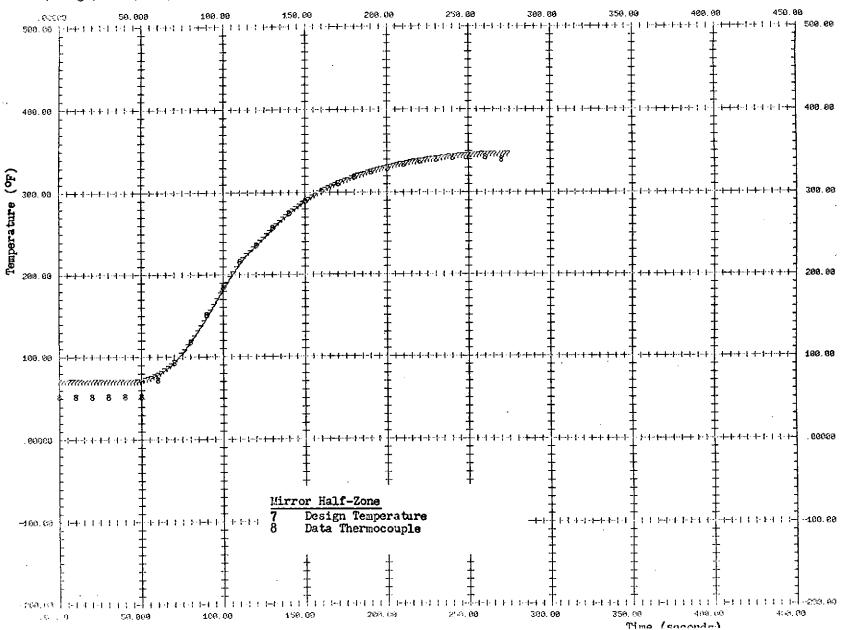


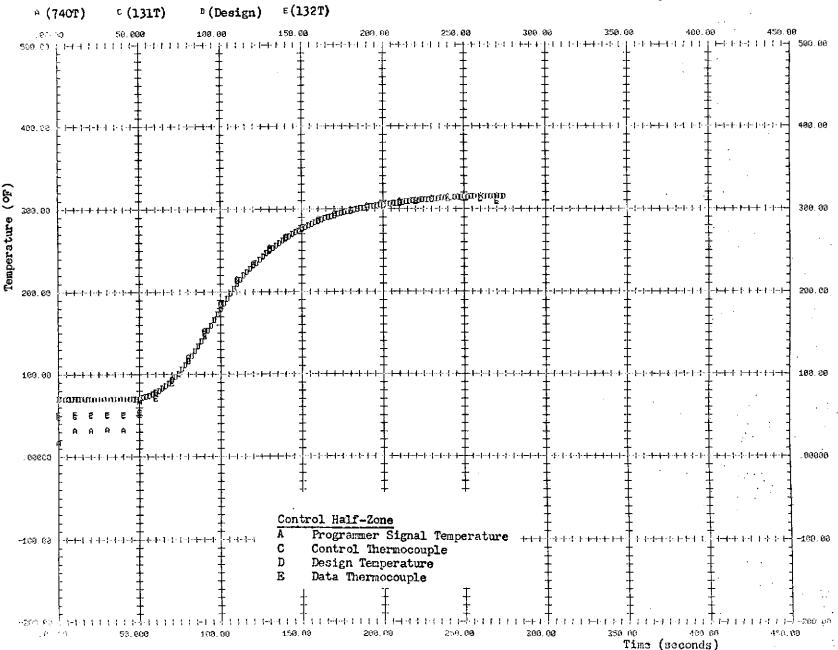














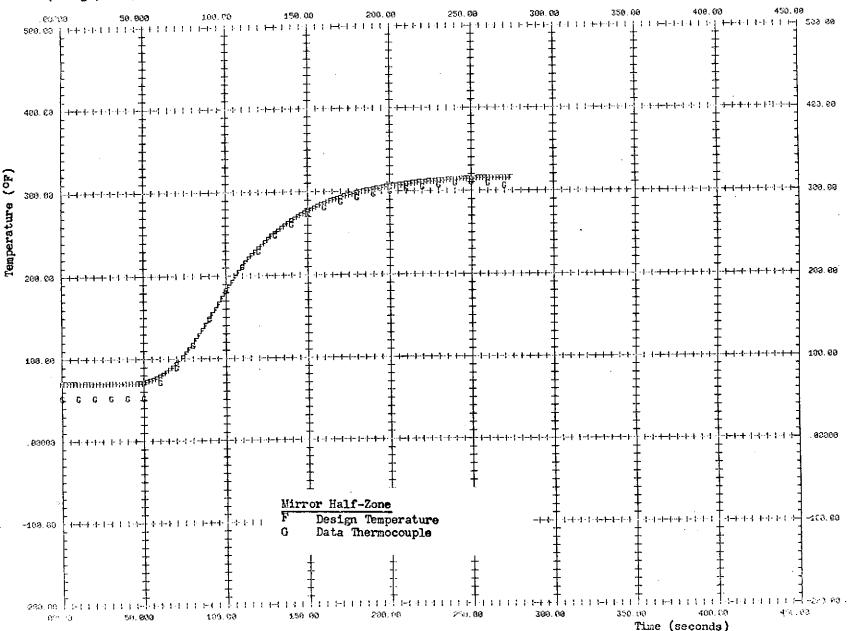
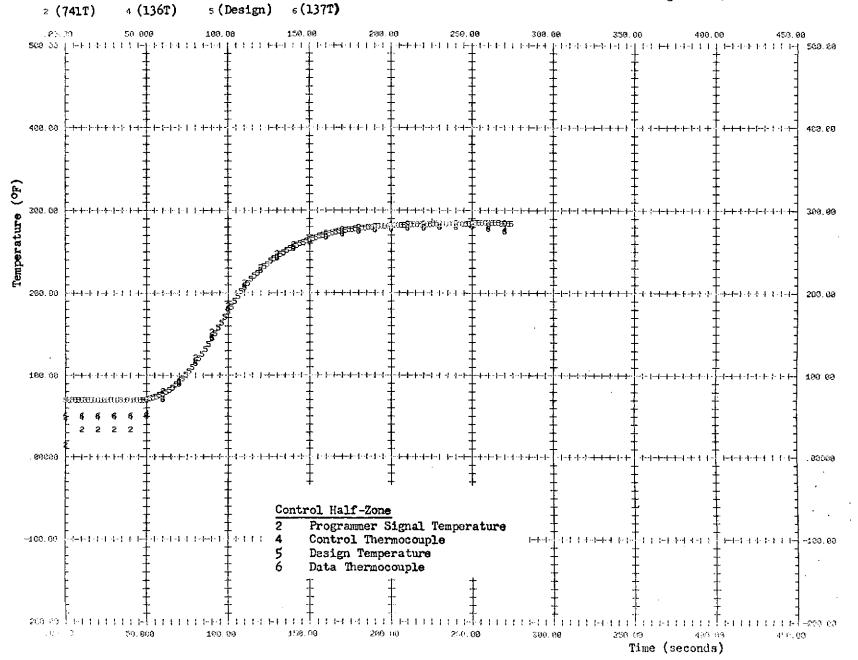
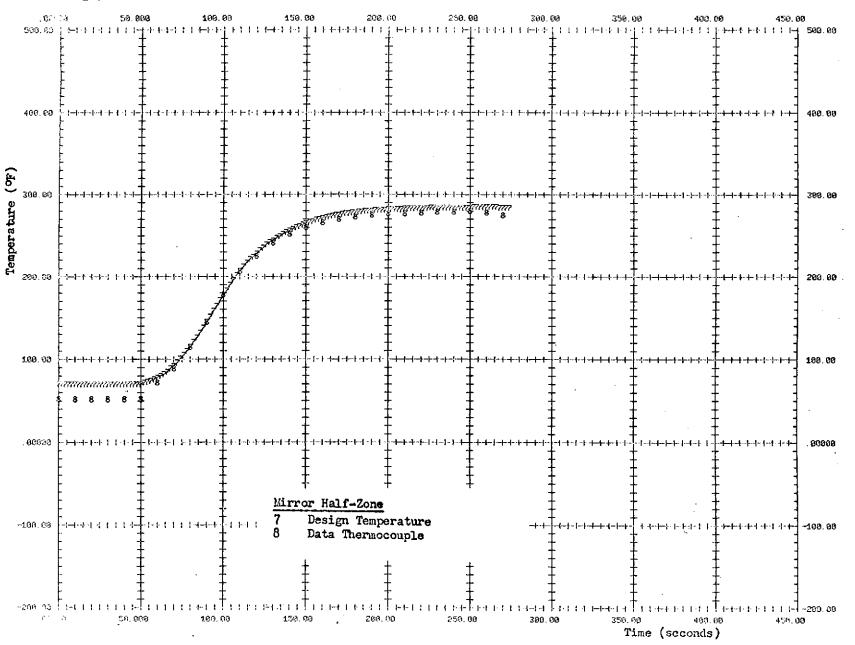
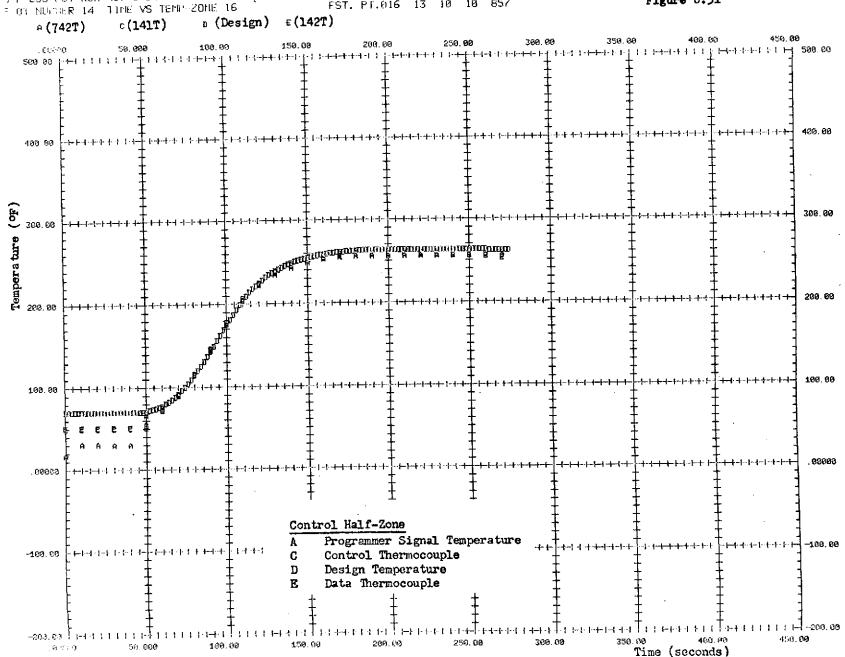


Figure 8.29

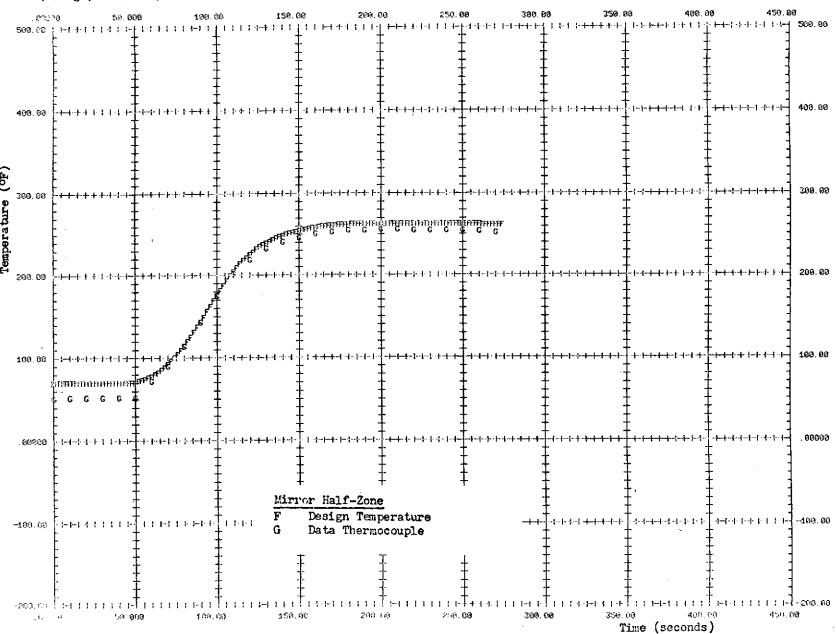


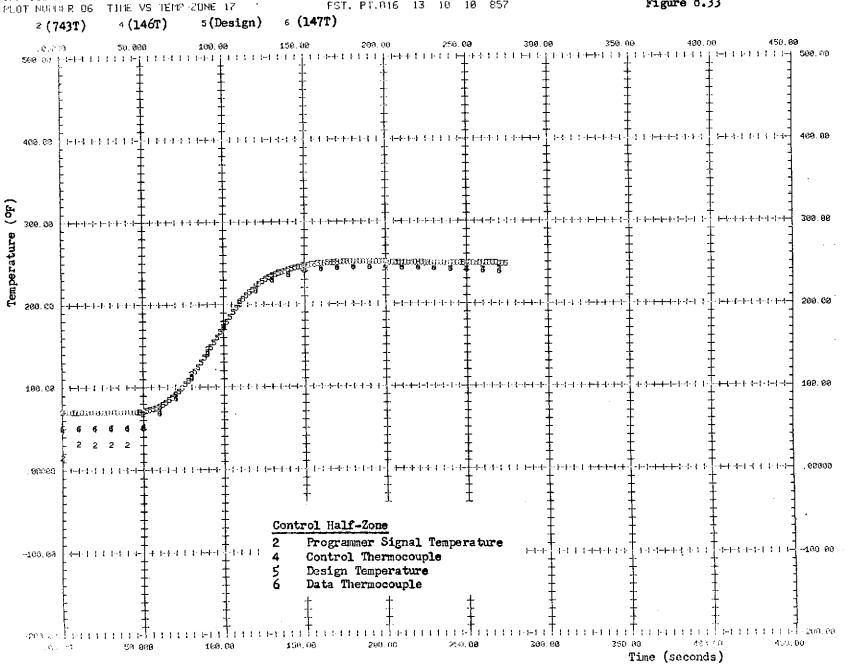




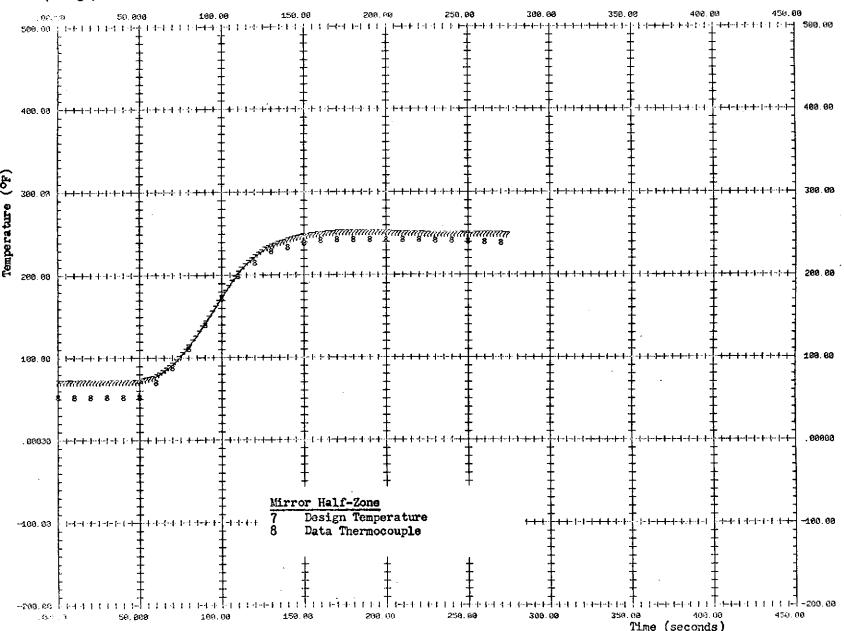


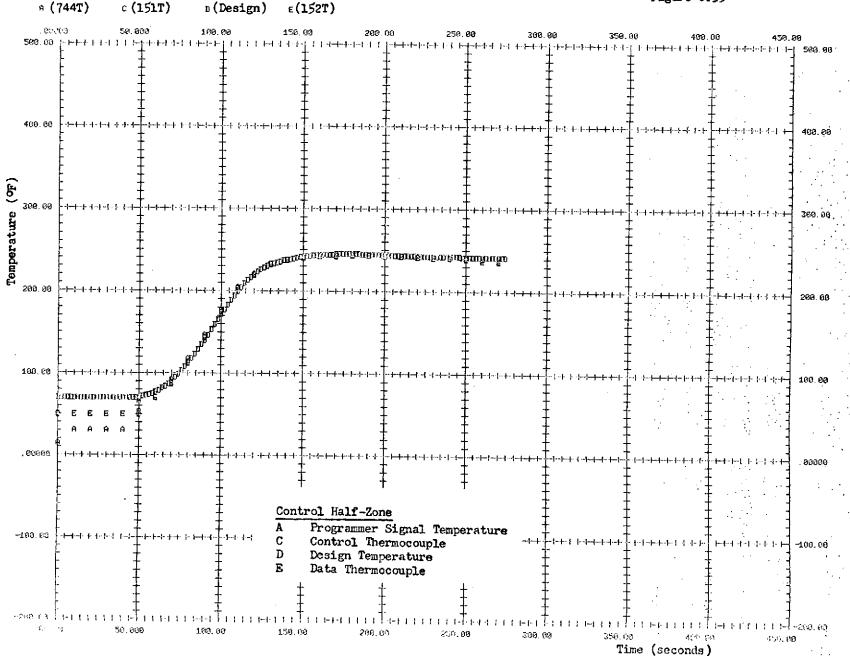
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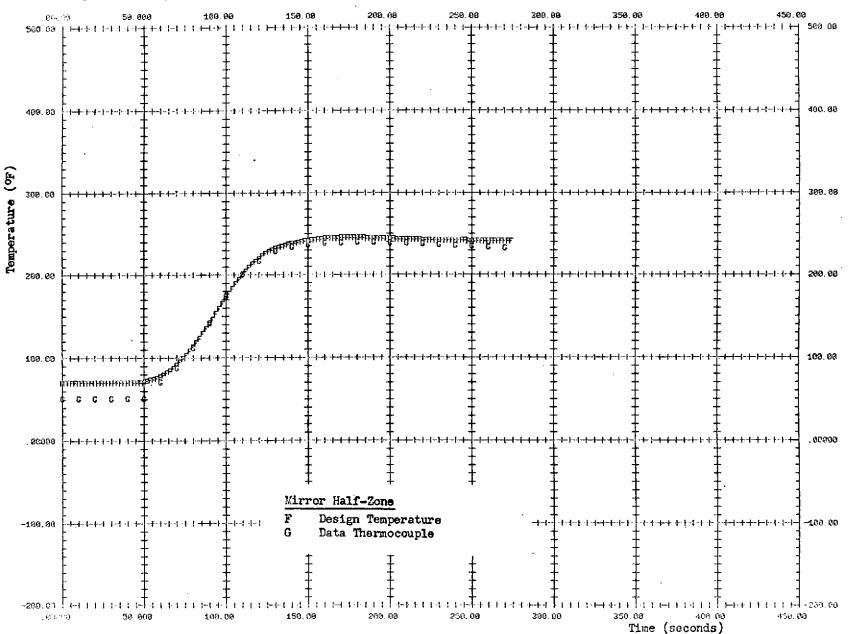
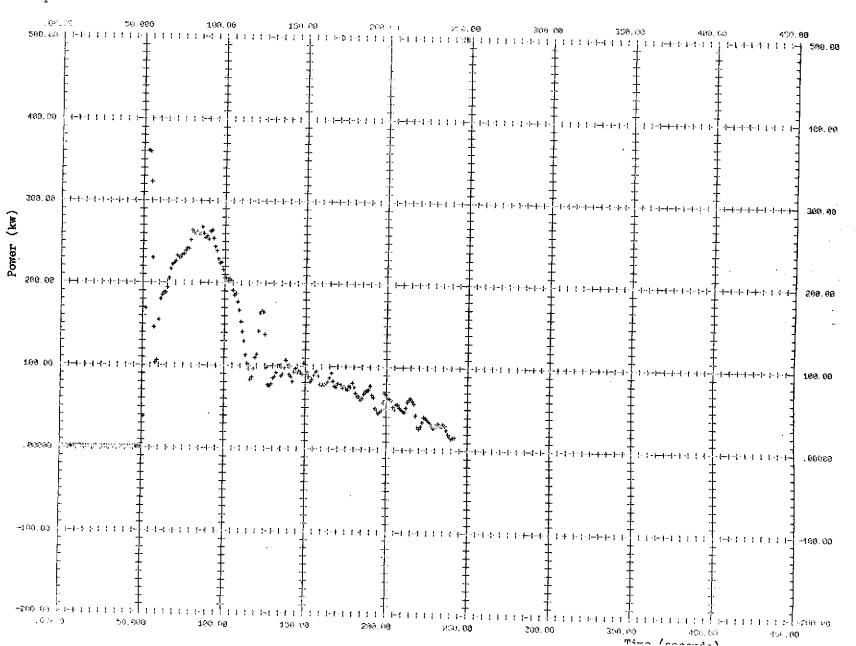


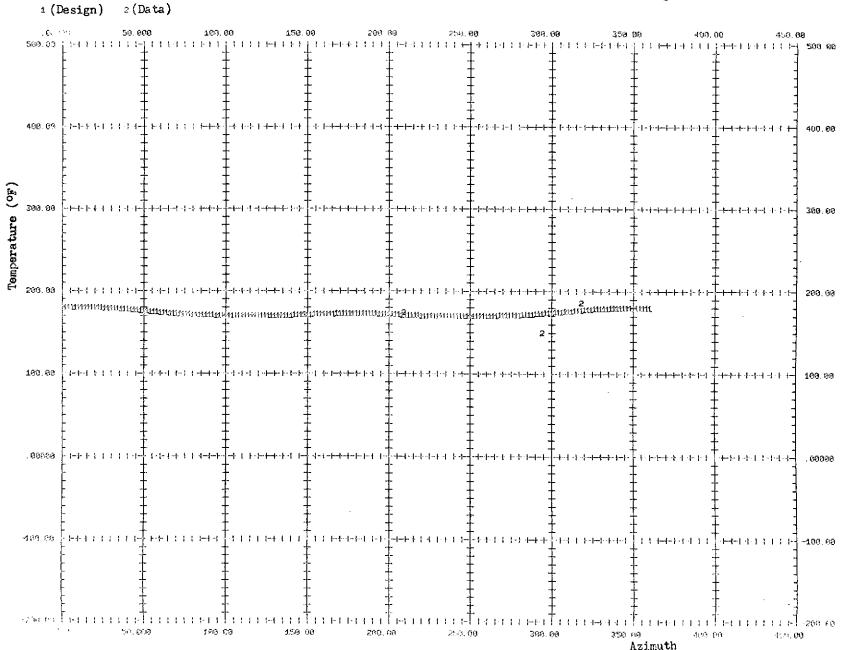
Figure 9.

Power history for heating zone no. 1 power controller.

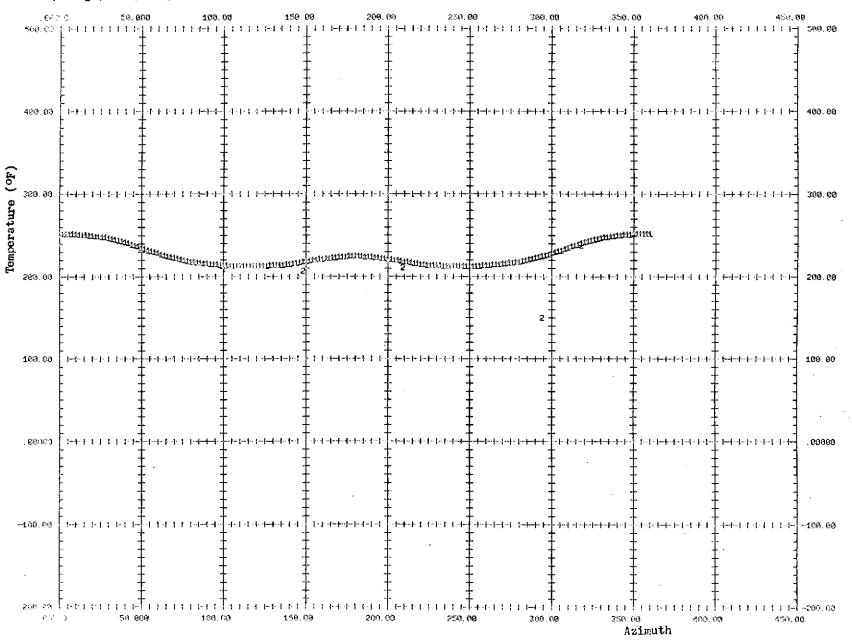


Figures 10.1 thru 10.40.

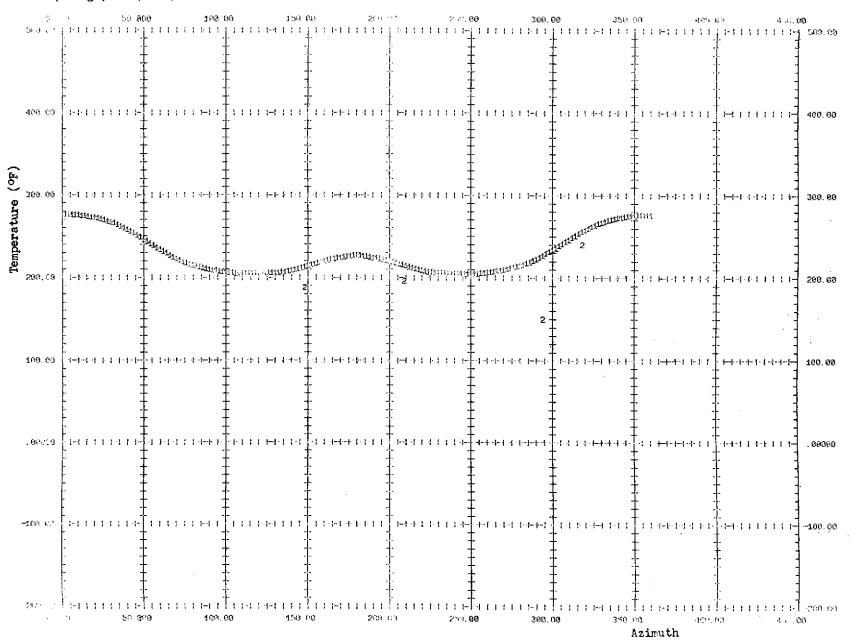
Design and data thermocouples circumferential temperature distributions.



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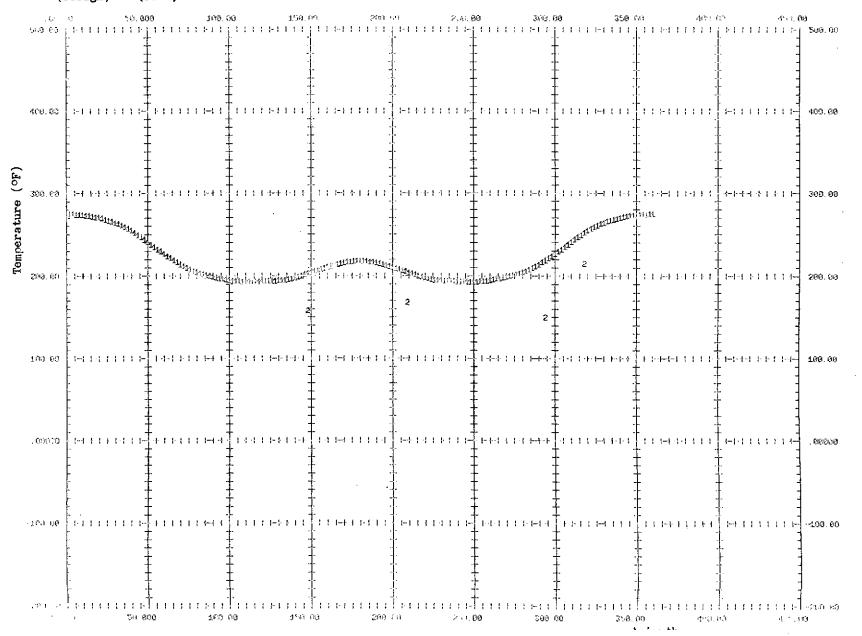
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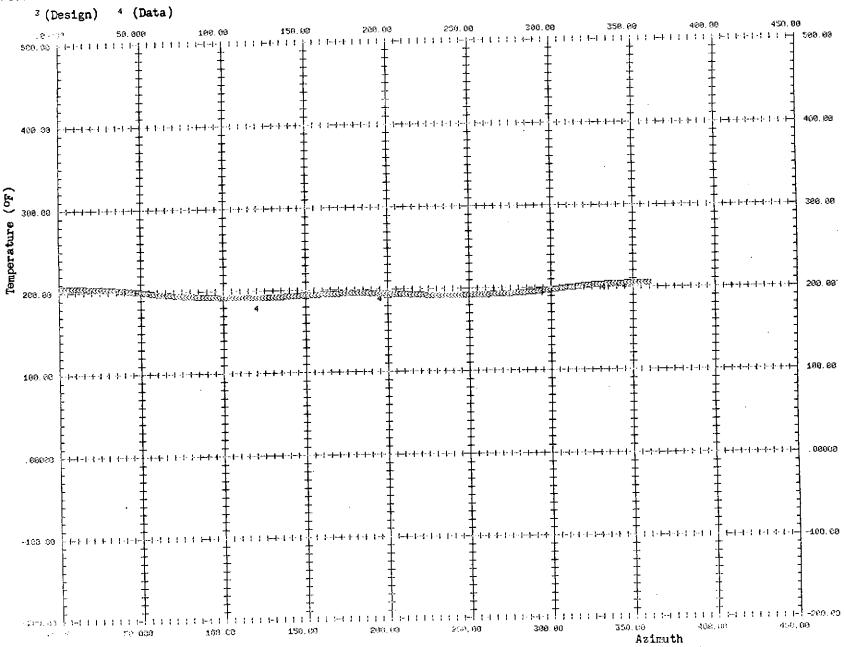
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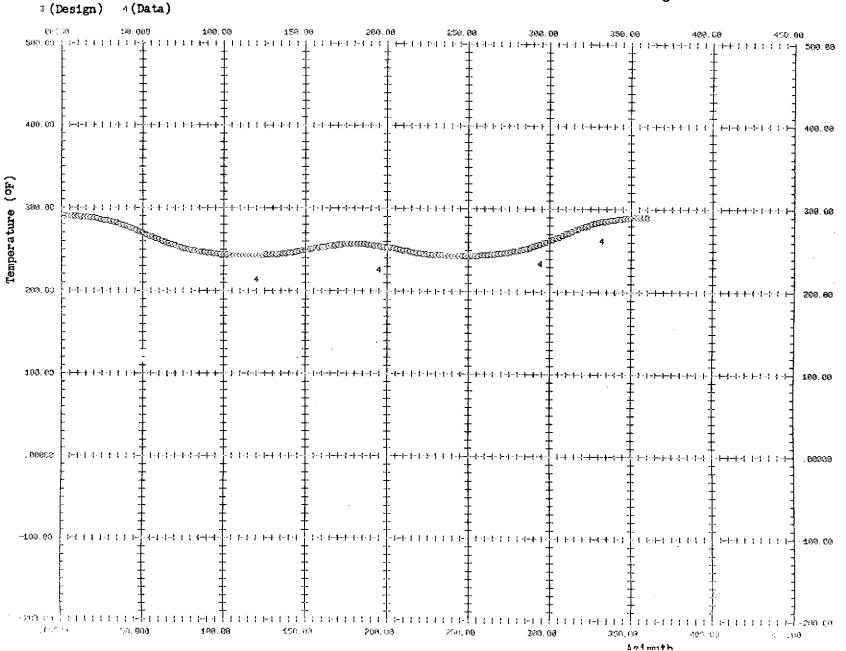
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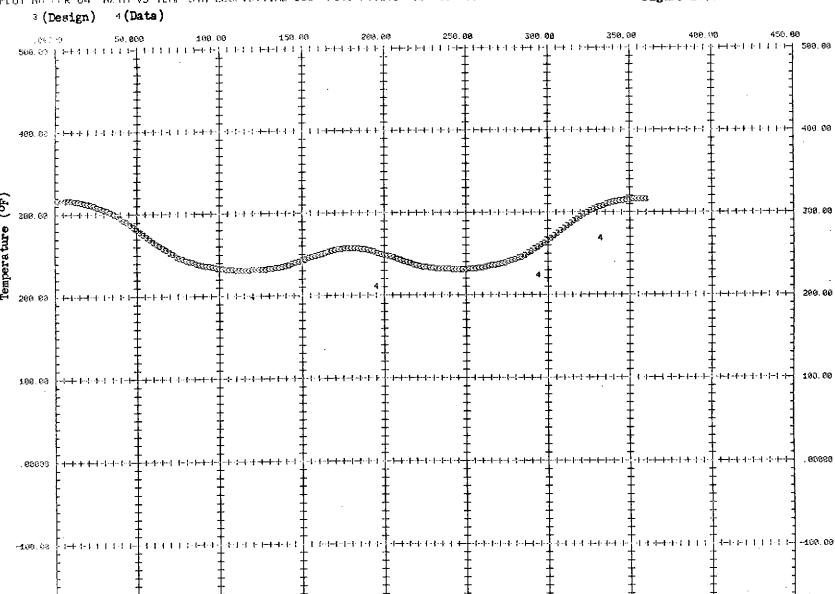
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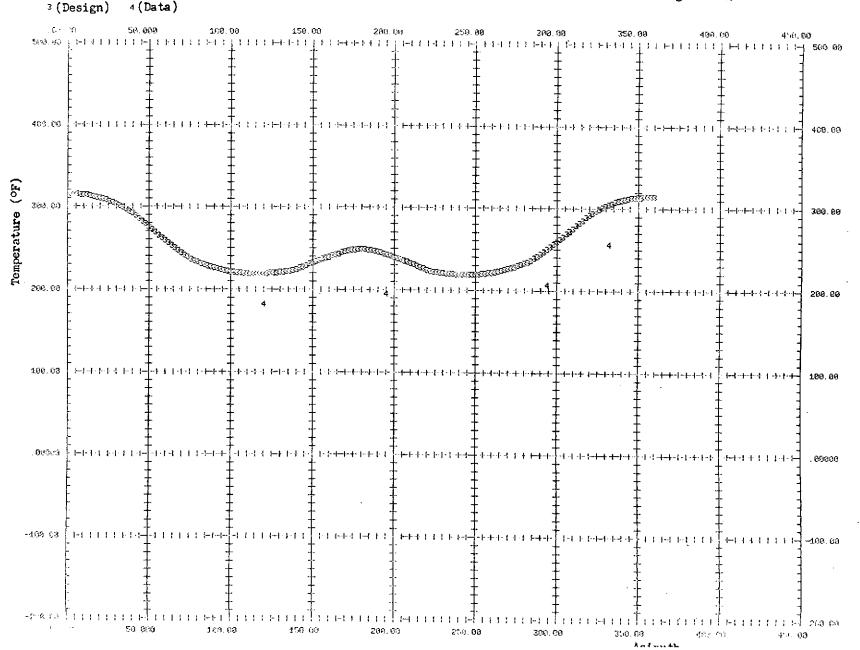
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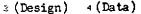
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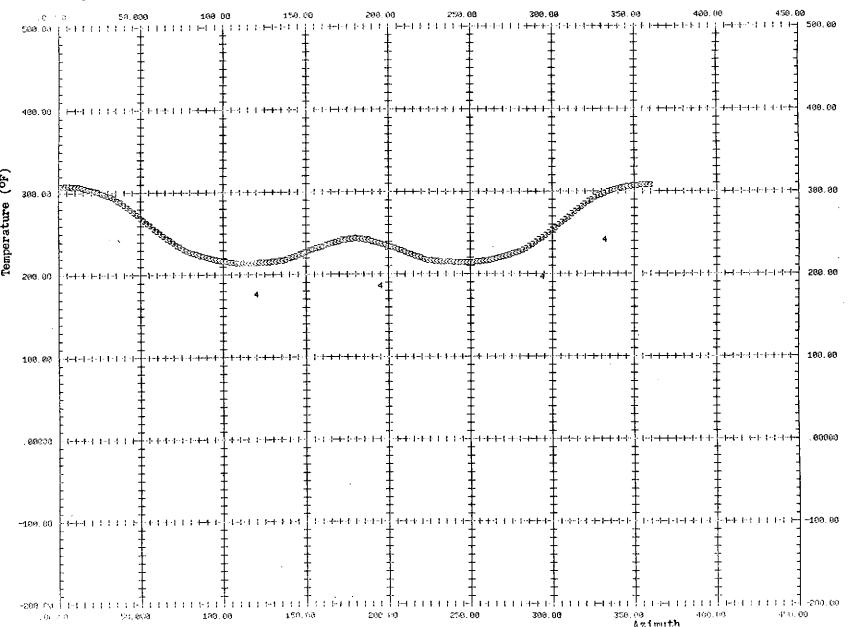
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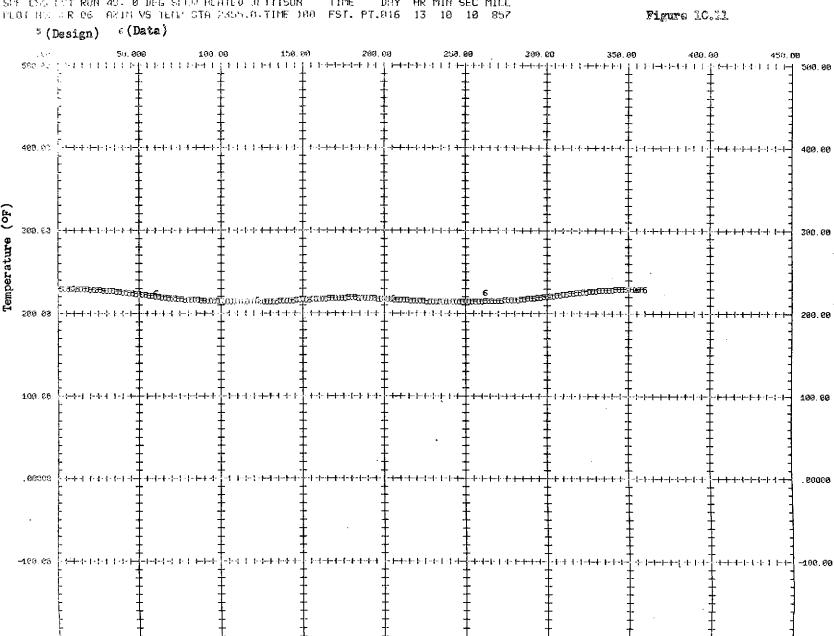
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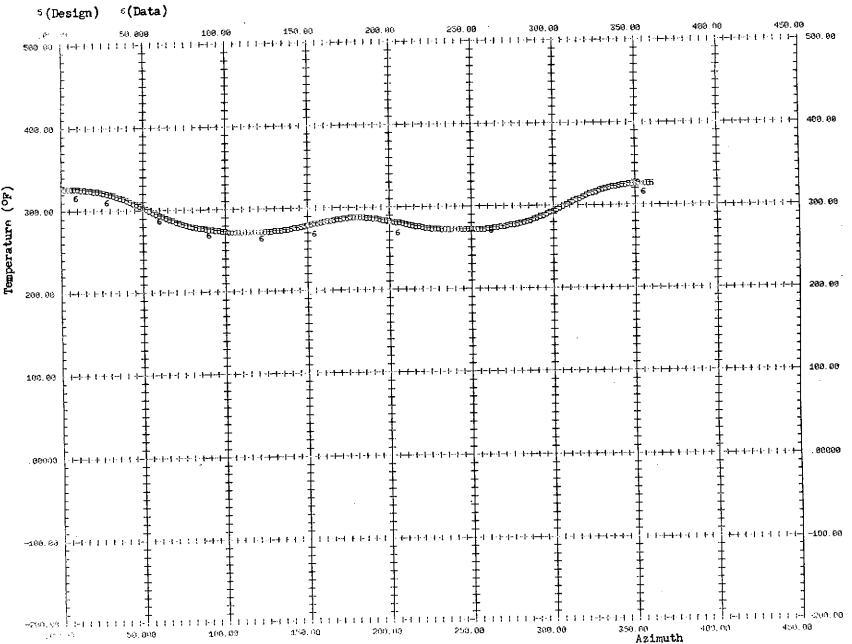
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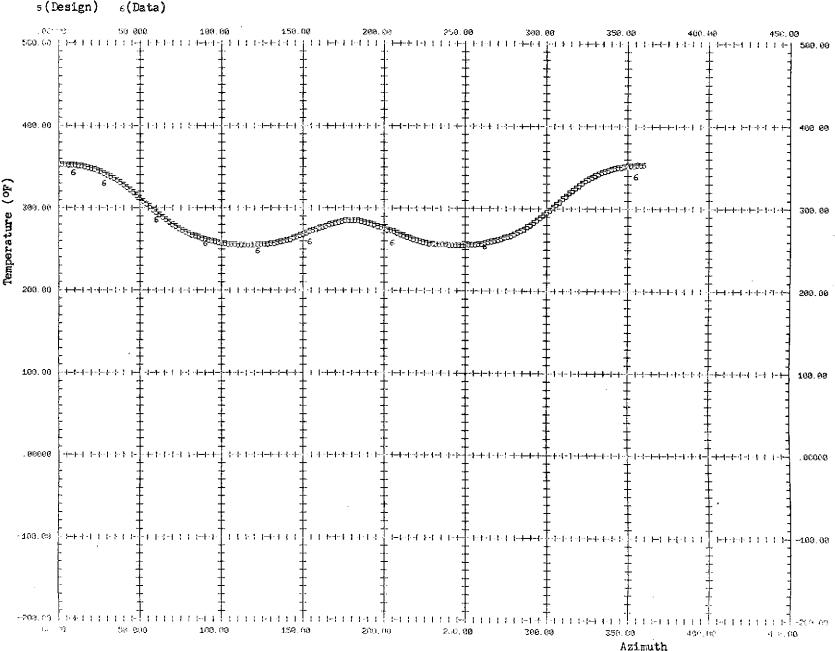
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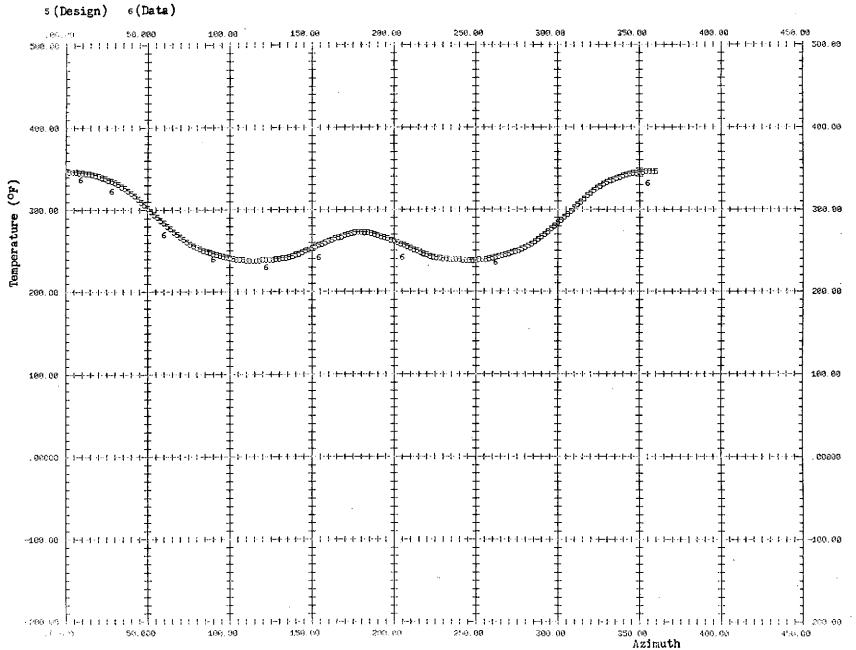
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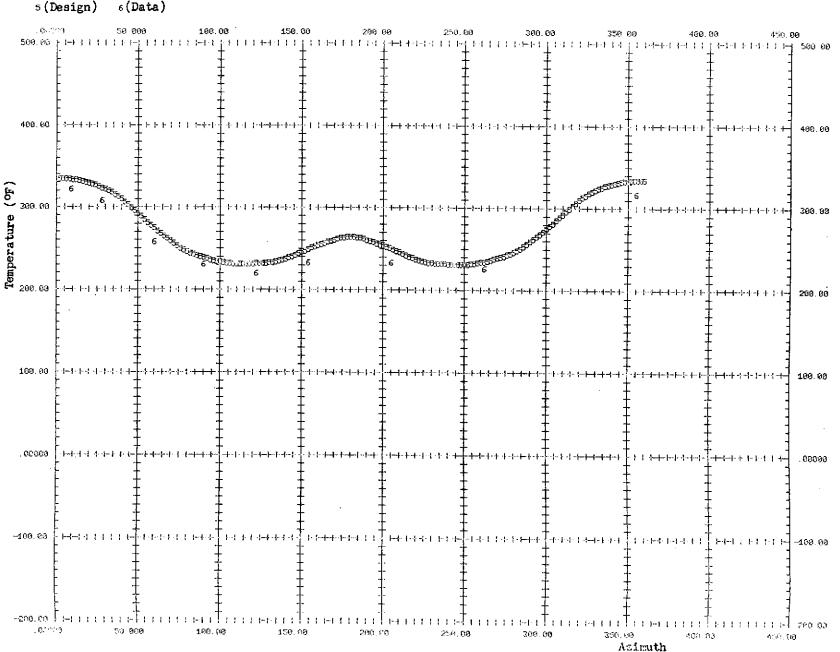
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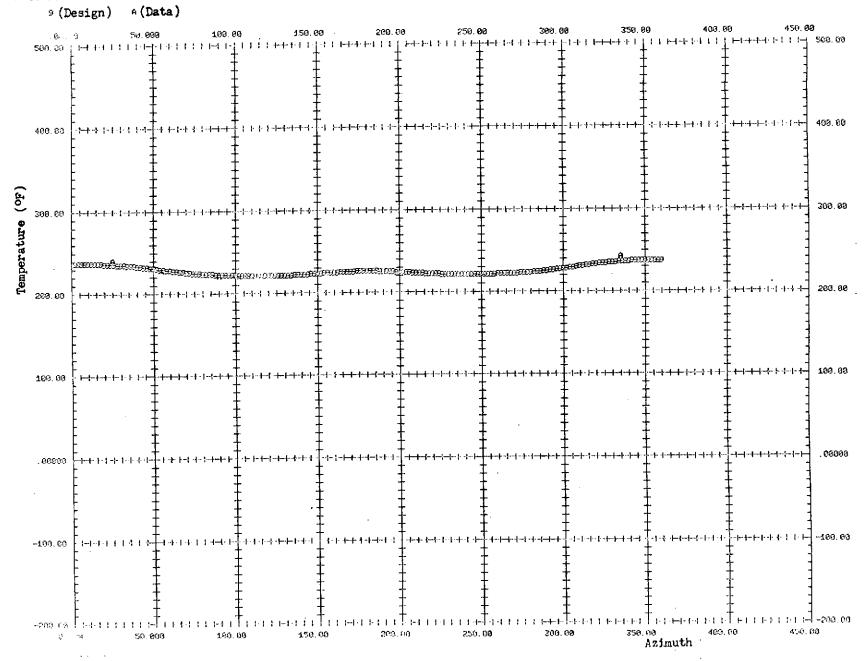
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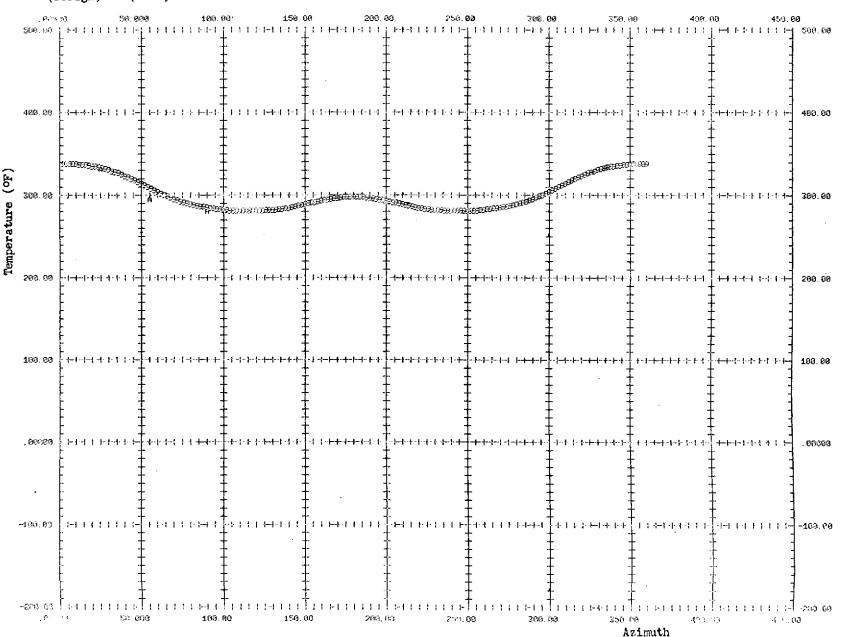








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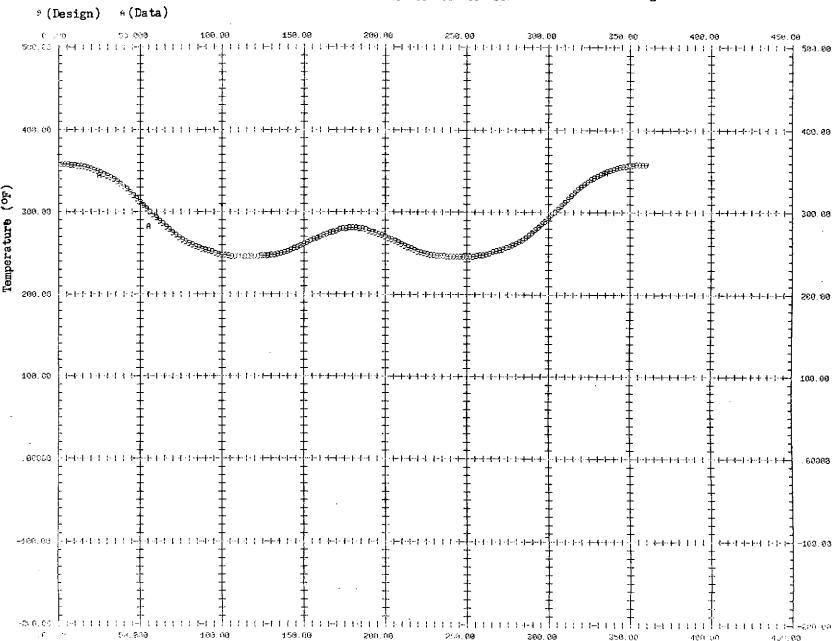
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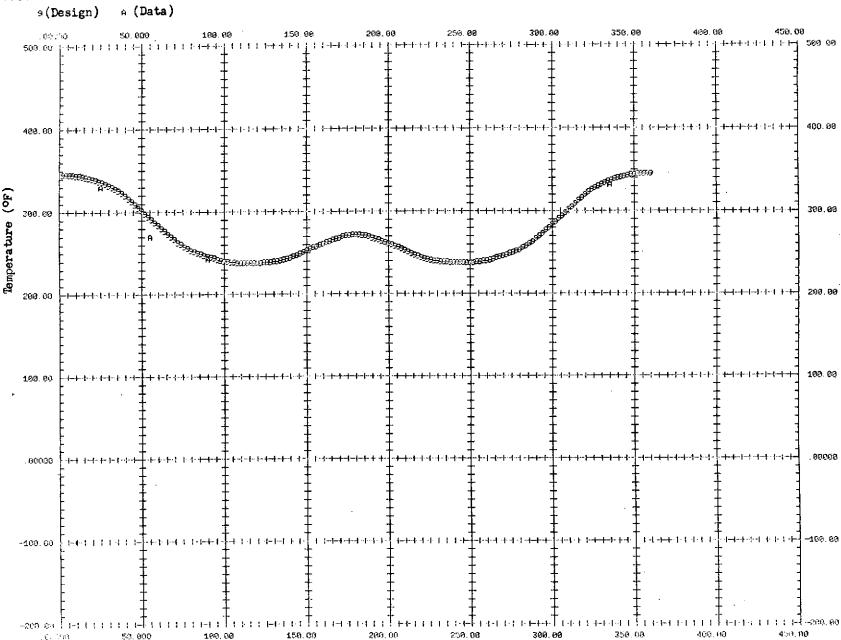
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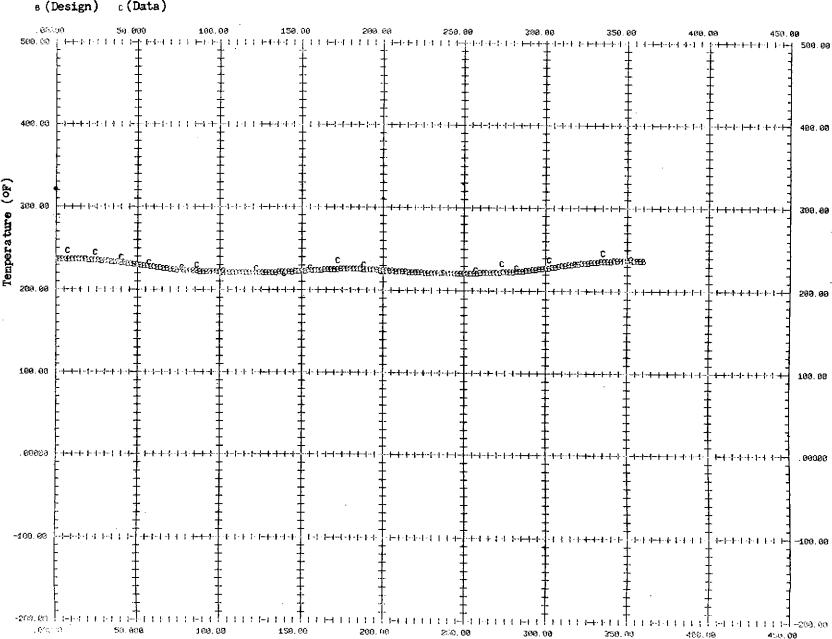
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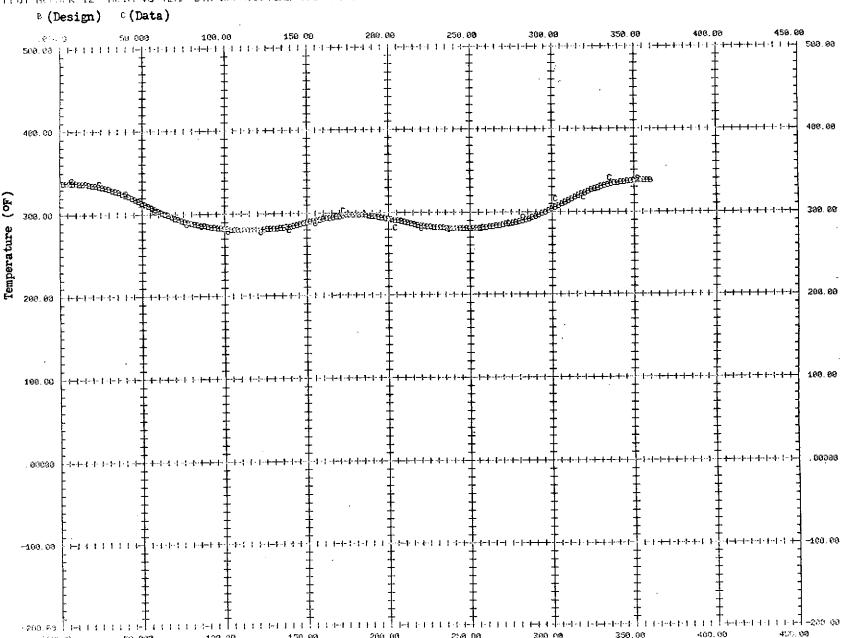
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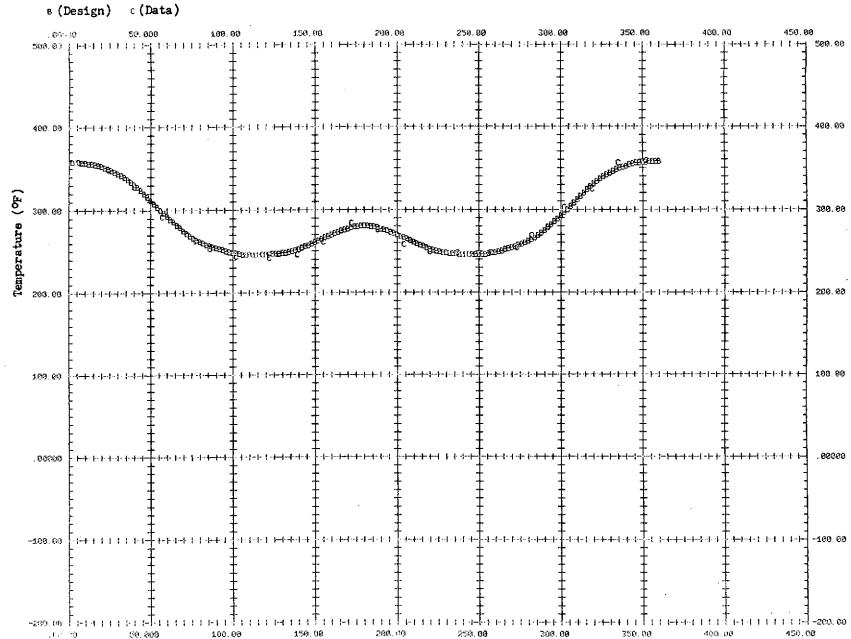
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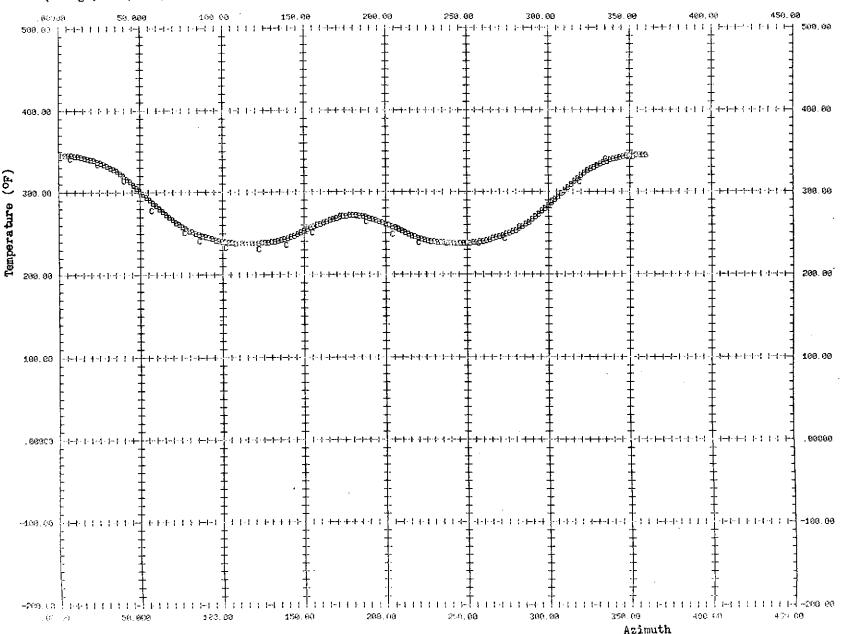
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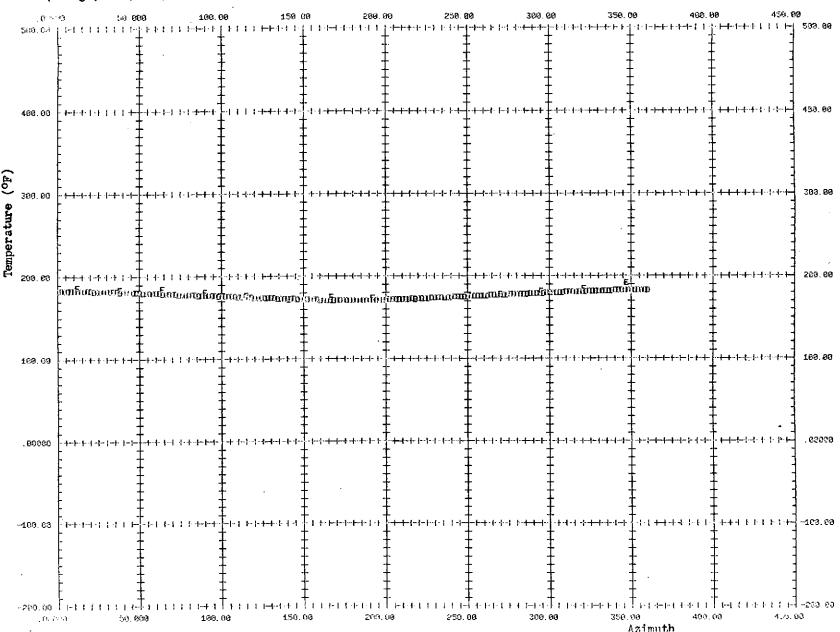
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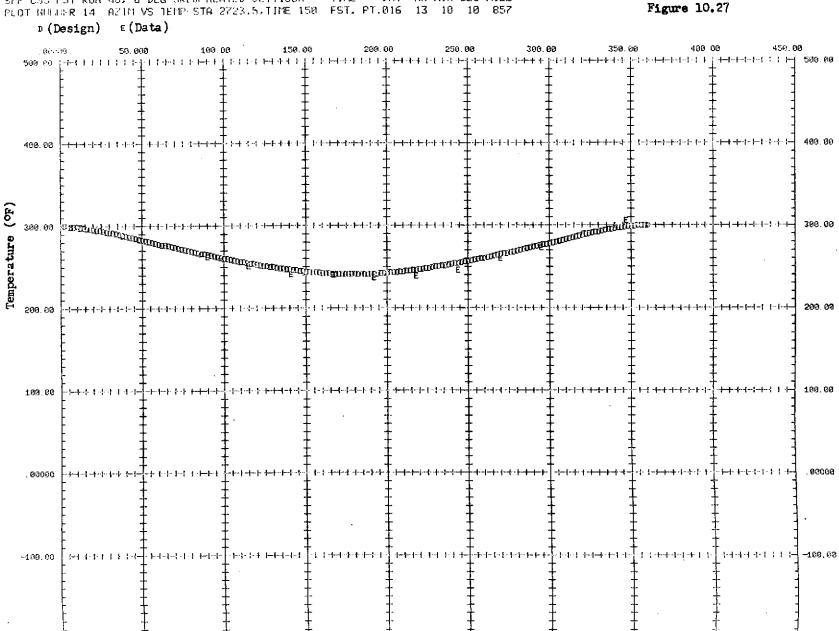
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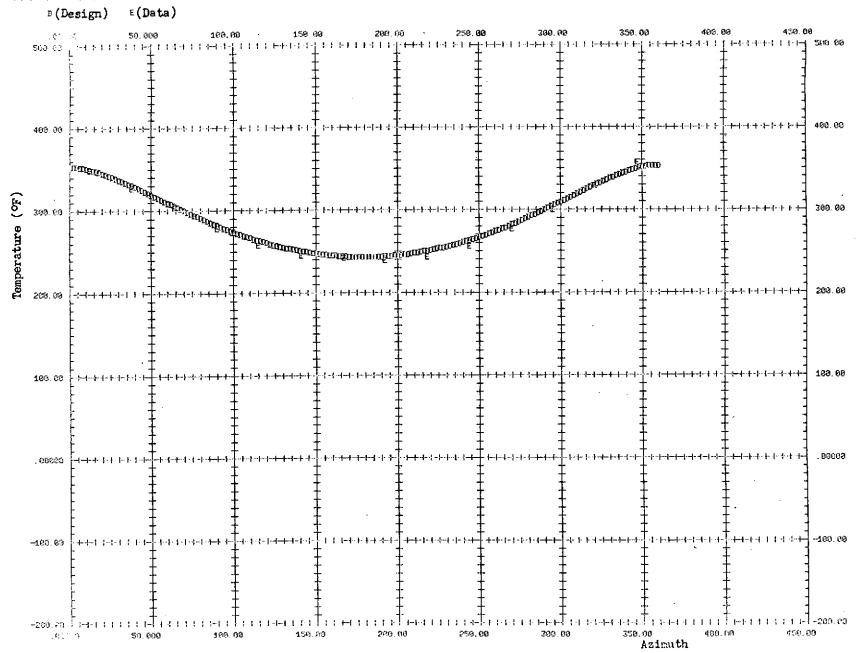
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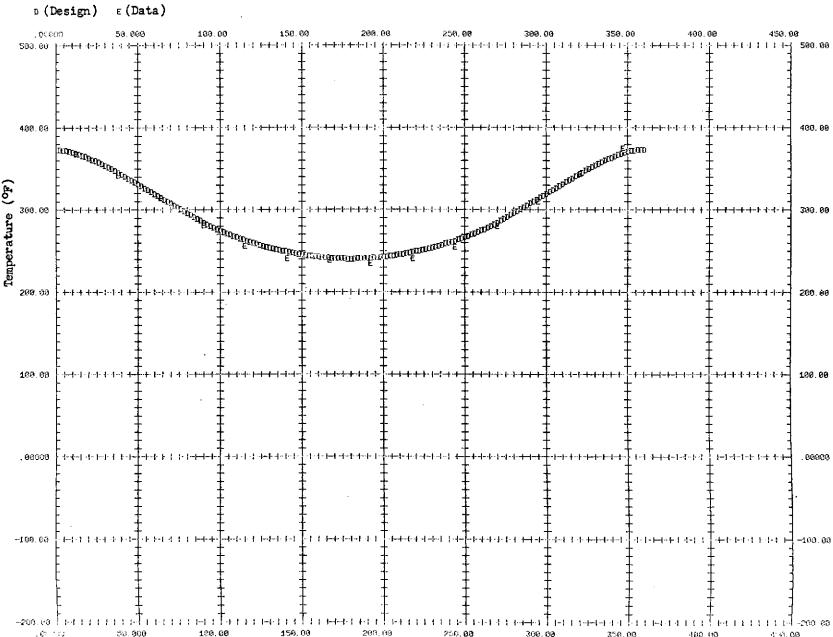
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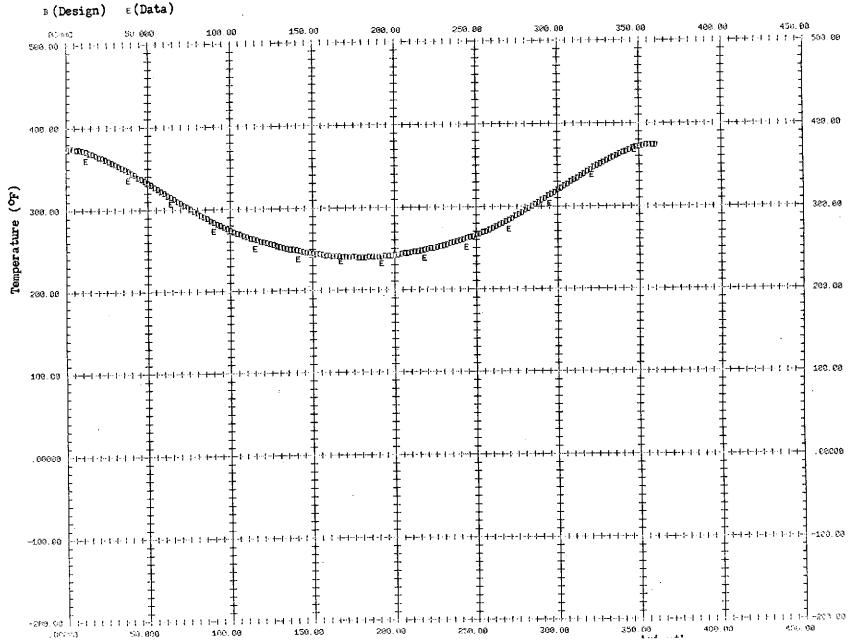
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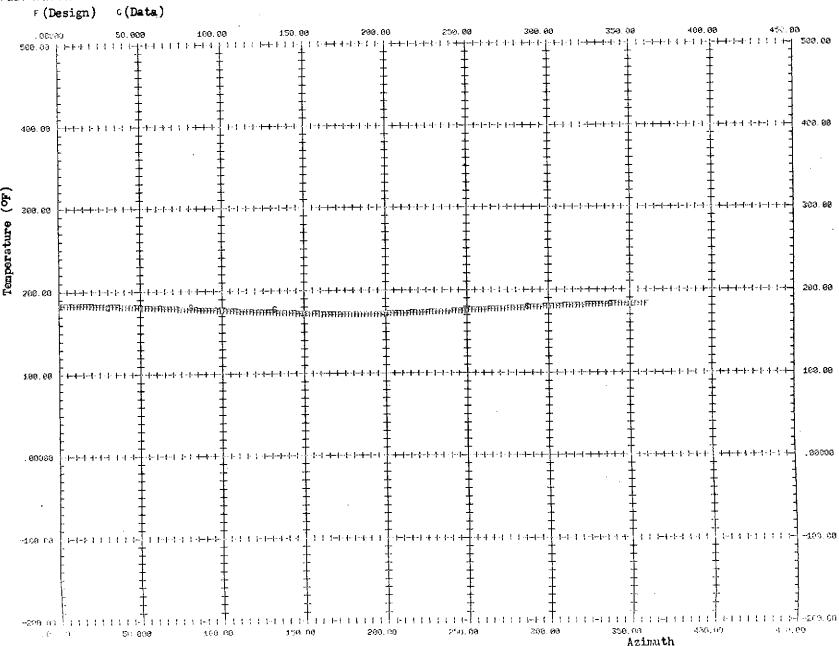
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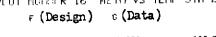


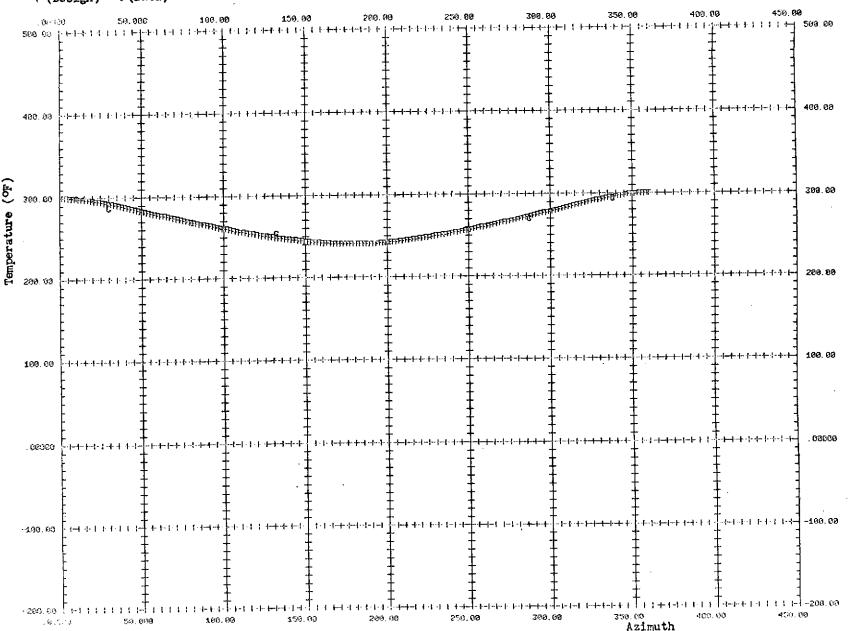












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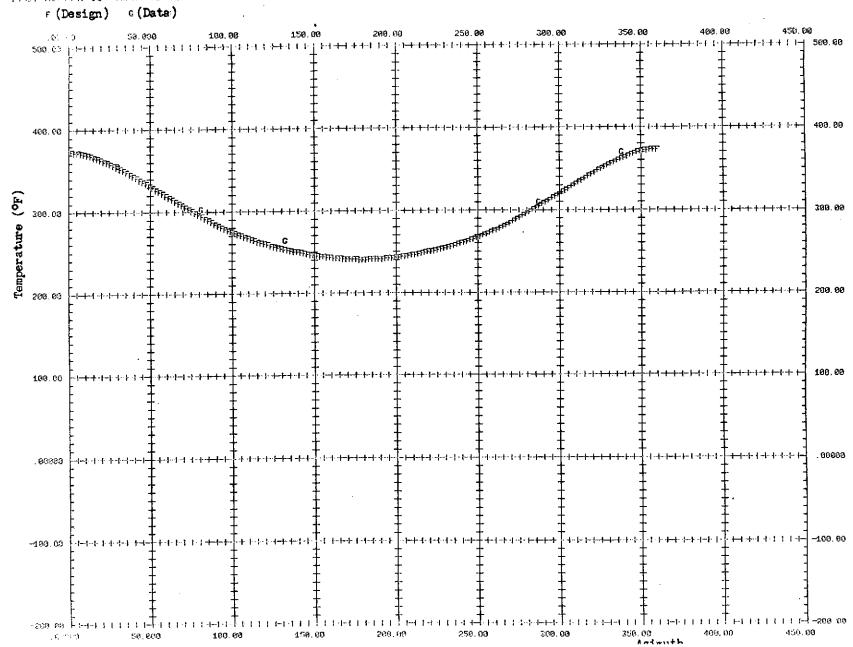
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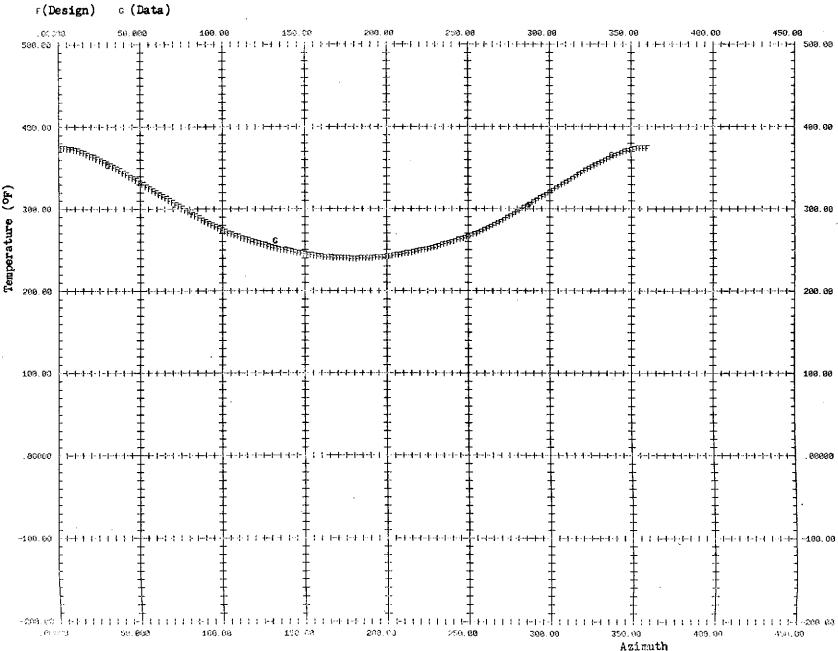
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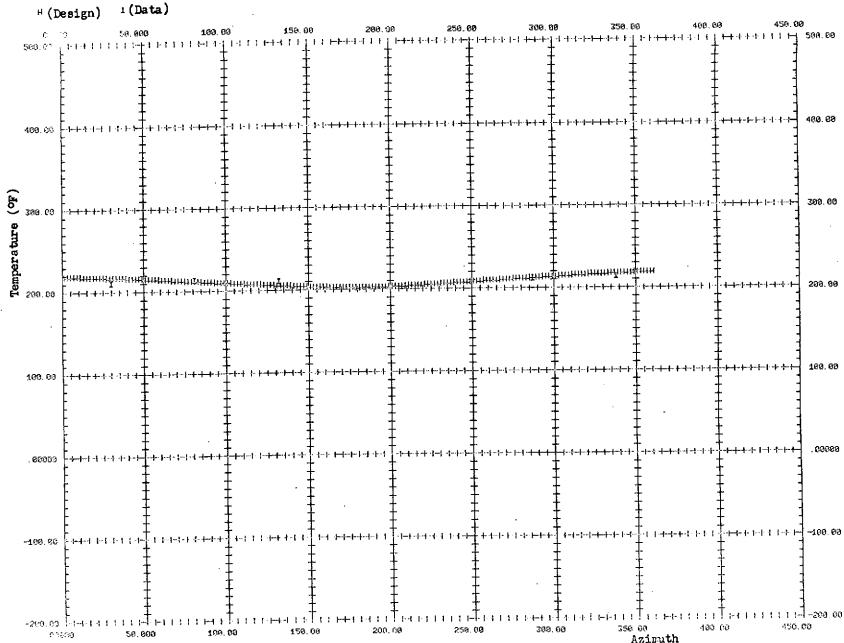
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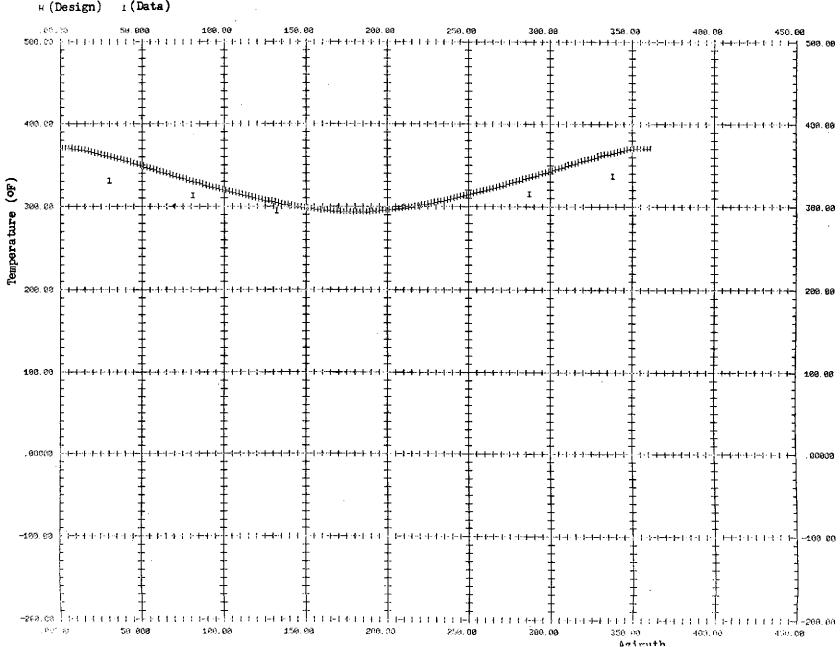
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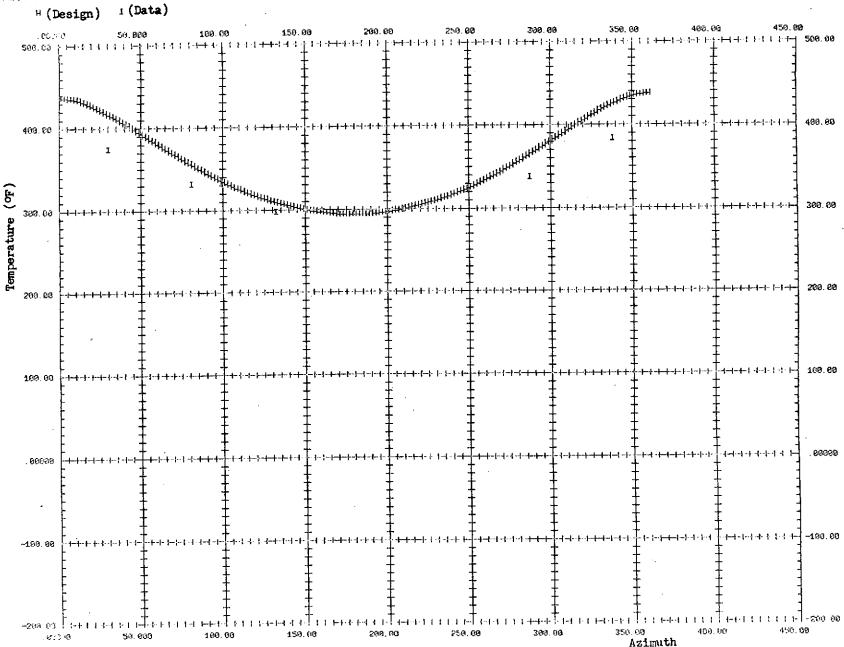
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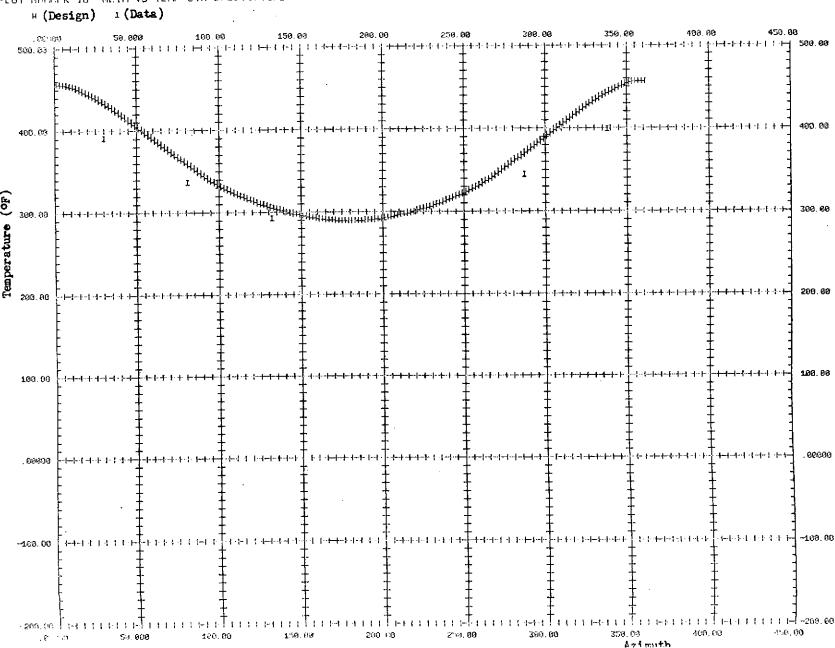


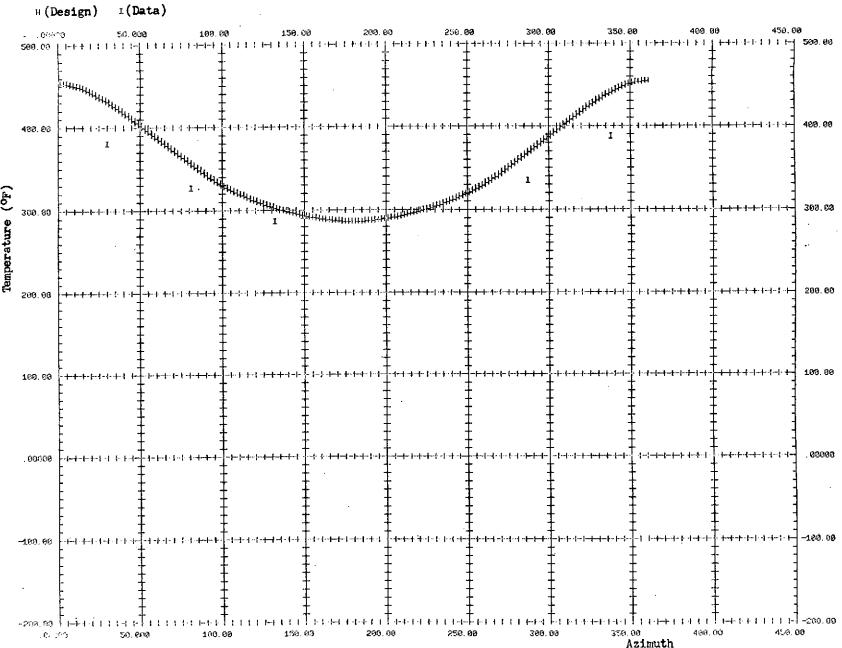






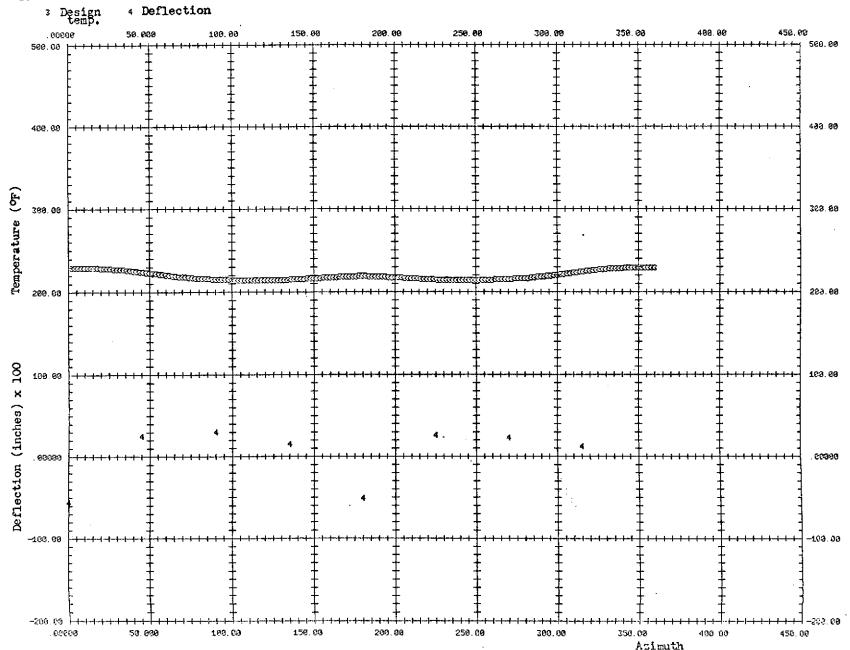


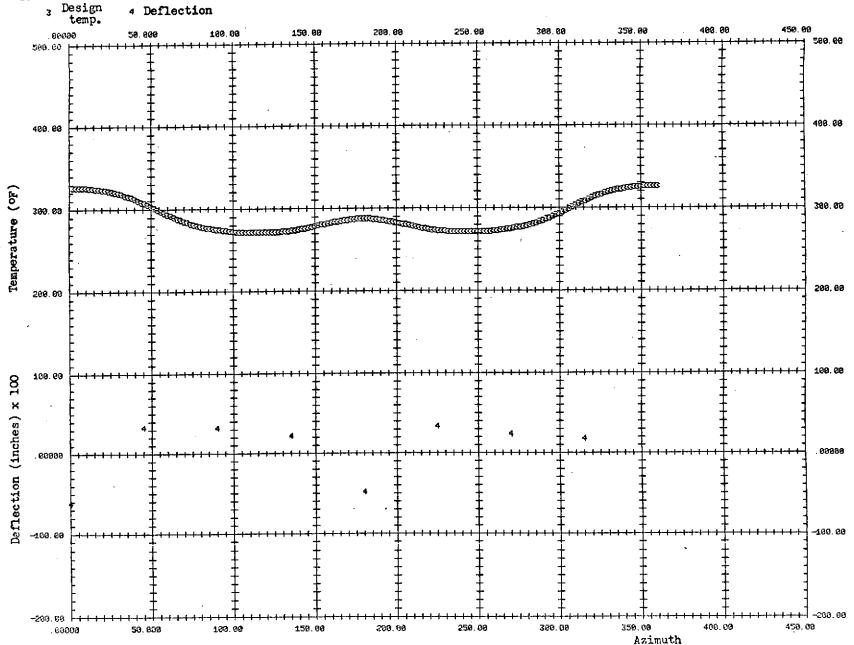




Figures 11.1 thru 11.15.

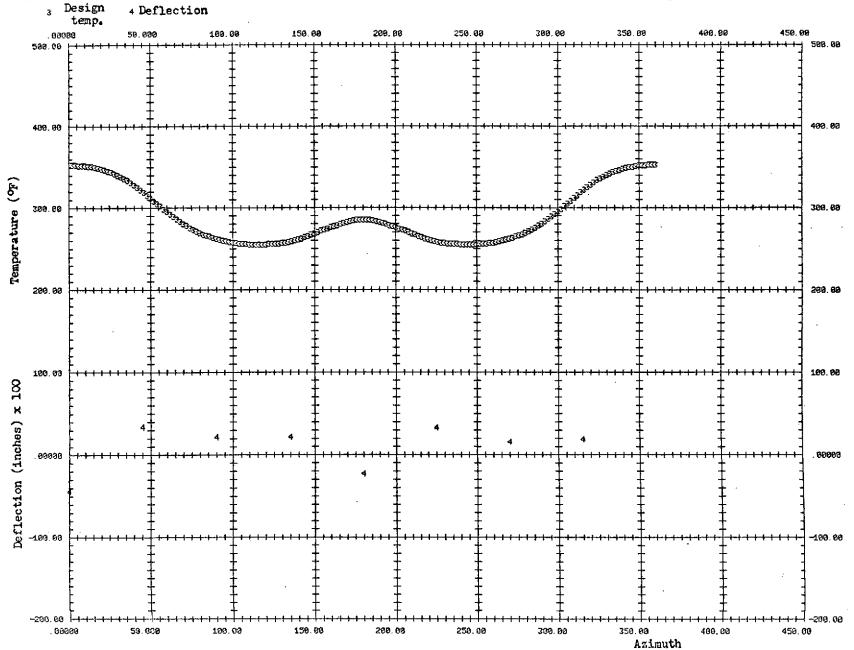
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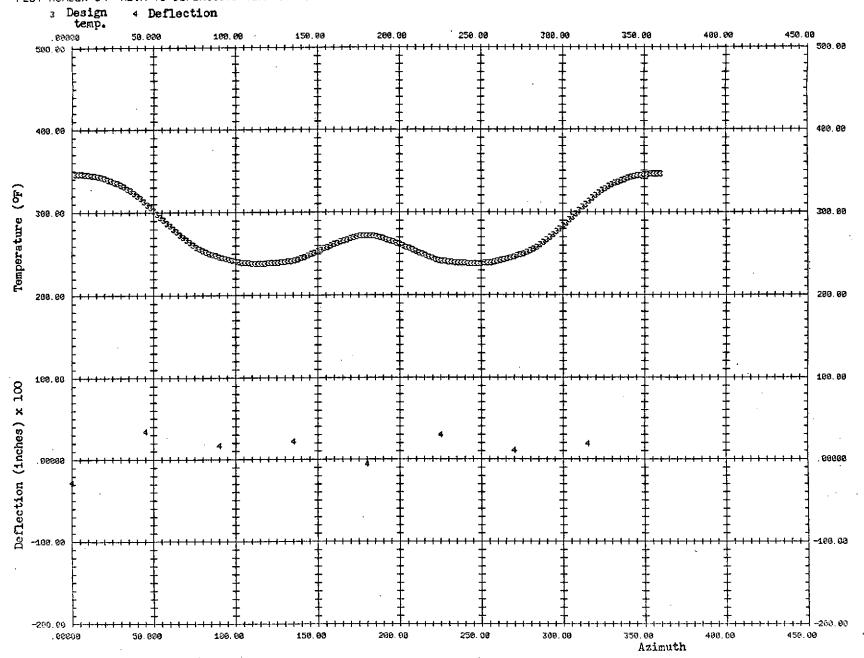


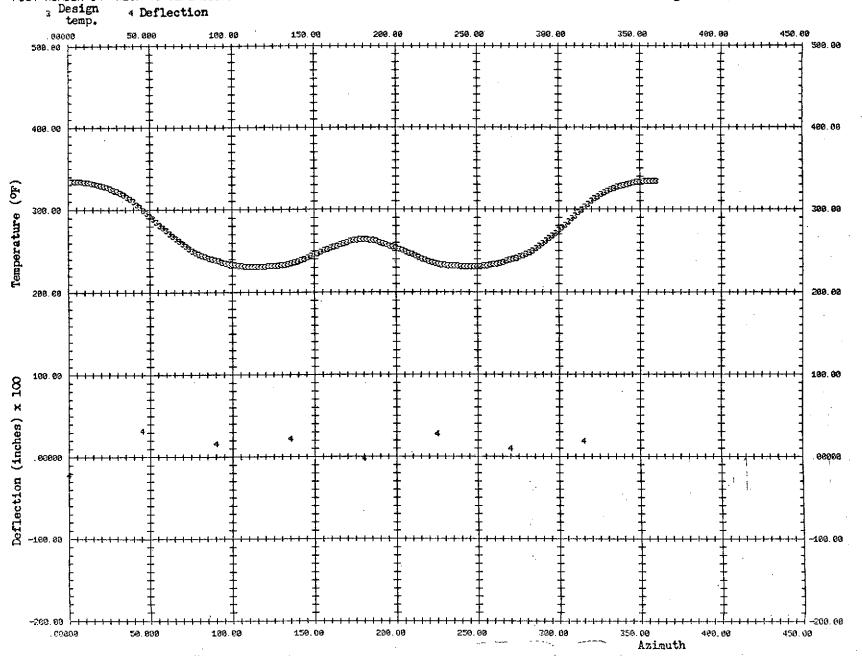


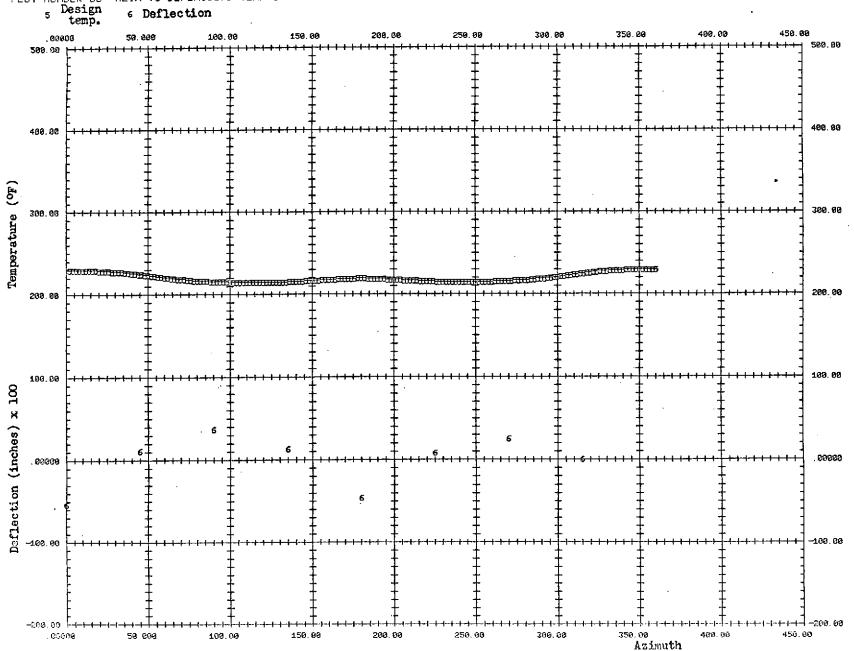
SPF CSS-FST RUN 48, 0 DEG SKEW HJT, TIME 200 SEC TIME DAY HR MIN SEC MILL PLOT NUMBER 04 AZIM VS DEFLX100,D TEMP-STA 2377 FST. PT.016 13 10 10 857

Figure 11.3



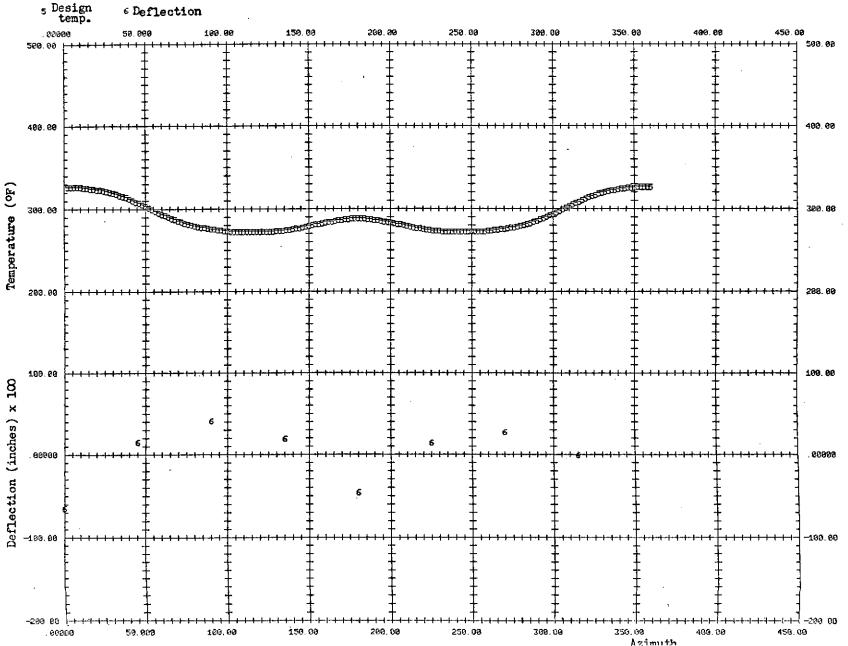


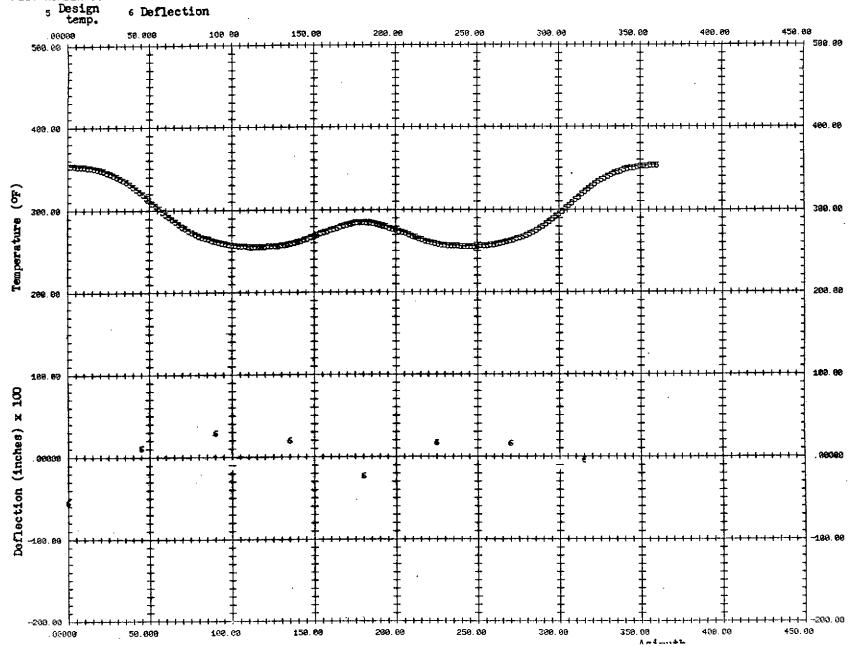




SPF CSS-FST RUN 48, Ø DEG SKEW HJT, TIME 150 SEC TIME DAY HR MIN SEC MILL FLOT NUMBER 06 AZIM VS DEFLXI00.D TEMP-STA 2459 FST. PT.016 13 10 10 857

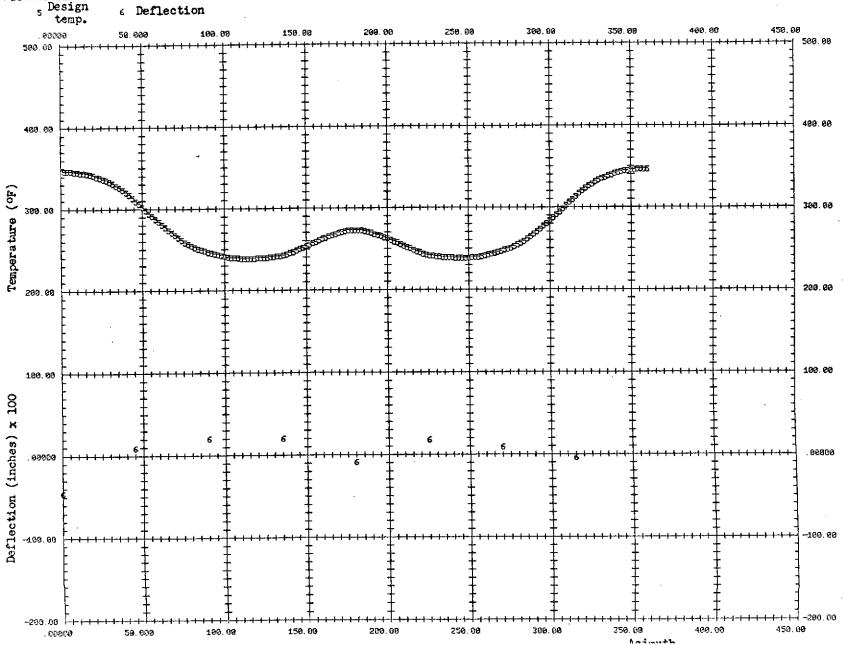
Figure 11.7

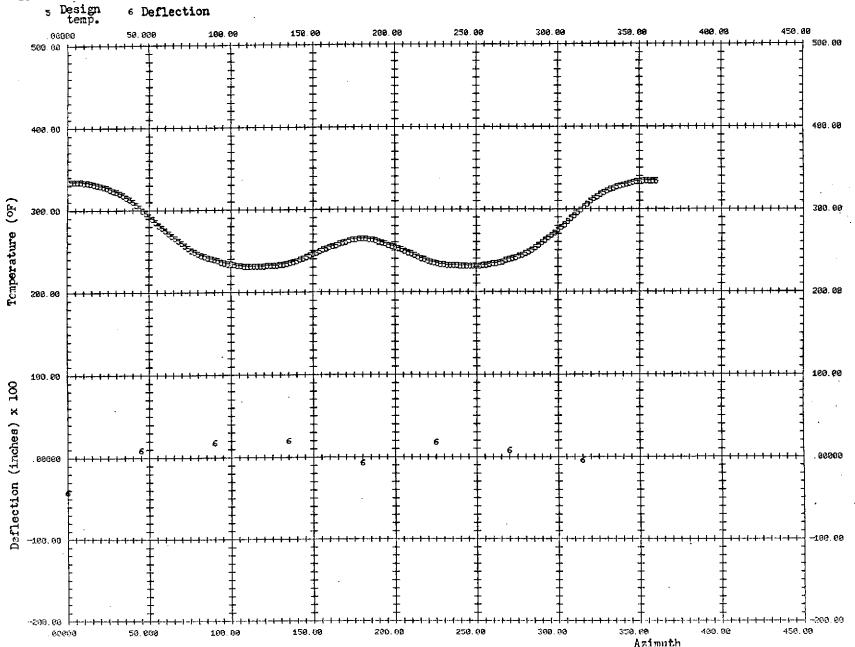




SPF CSS-FST RUN 48. Ø DEG SKEW HJT. TIME 250 SEC TIME DAY HR MIN SEC MILL PLOT NUMBER 06 AZIM VS DEFLX100.D TEMP-STA 2459 FST. PT.016 13 10 10 857

Figure 11.9





SPF CSS-FST RUN 48, Ø DEG SKEW HJT, TIME 100 SEC TIME DAY HR MIN SEC MILL PLOT NUMBER 12 AZIM VS DEFLX100.D TEMP-STA 2664 FST. PT.016 13 10 10 857

Figure 11.11

